

ISFNT15

10-15  
SEPT 2023  
AUTORING  
ALFONSO KEELER  
ENI PROMISSE ON  
ORSH CAMPANA  
SMAN



# A Versatile Divertor for the Italian Divertor Tokamak Test Facility

*S. Rocella*

*R. Neu, P. Innocente, R. Ambrosino, A. Castaldo, C. Day, J. Gunn, H. Greuner, K. Hunger  
D. Marzullo, G.M. Polli, A. Reale, H. Roche, C. Tantos, N. Vignal &  
In-Vessel Component Team*



**ENEA**

Italian National Agency for New Technologies,  
Energy and Sustainable Economic Development



**IPP**

Max-Planck-Institut  
für Plasmaphysik



**CONSORZIO REX**  
Ricerca Formazione Istruzione



**cea**



**POLITECNICO  
MILANO 1863**



**KIT**  
Karlsruhe Institute of Technology



This work has been carried out within the framework of the EUROfusion Consortium and has received funding from the Euratom research and training programme 2014-2018 and 2019-2020 under grant agreement No 633053. The views and opinions expressed herein do not necessarily reflect those of the European Commission.

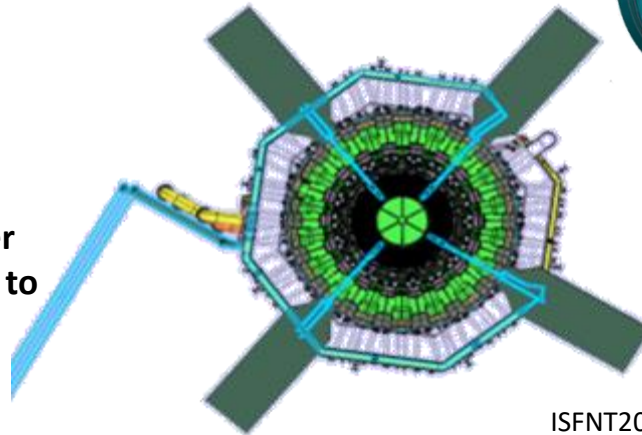
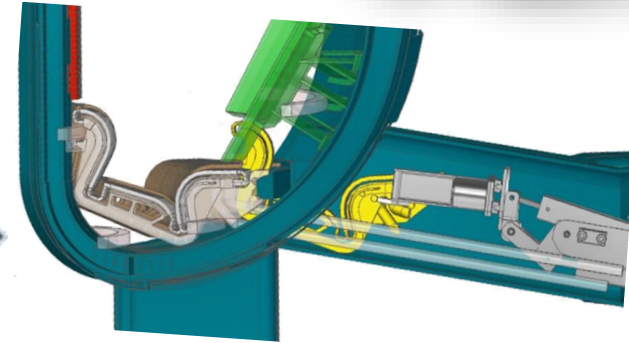
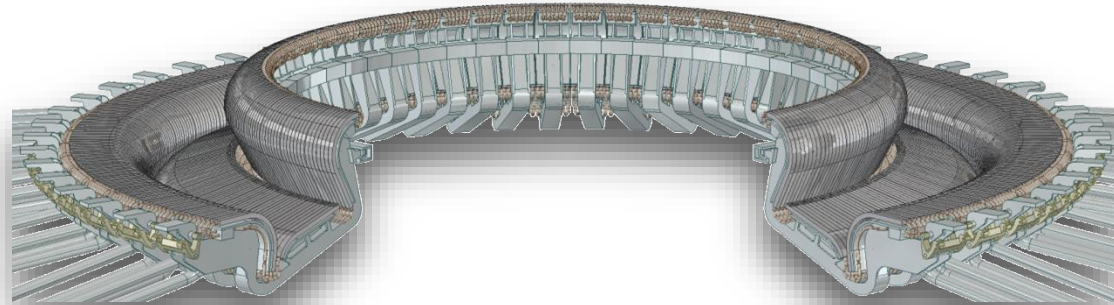
The **Divertor** consists of:  
**54** actively water-cooled **modules**  
(3 for each 20° sector) and supplied in parallel.

### Water Cooling System:

- inlet pressure: **5MPa**
- total mass flow rate up to: **577 kg/s**  
(**11m/s in the PFU pipes**)
- inlet temperature: **within 30-130°C**

The **4 central cassettes** in the RH ports are devoted to testing materials and technologies.

This central **cassettes** have to be easily **replaced** (without removing adjacent cassettes and piping) and can be supplied by a **dedicated water cooling system** (**43 kg/s; up to 250°C, up to 15 MPa**)



The Divertor consists of:

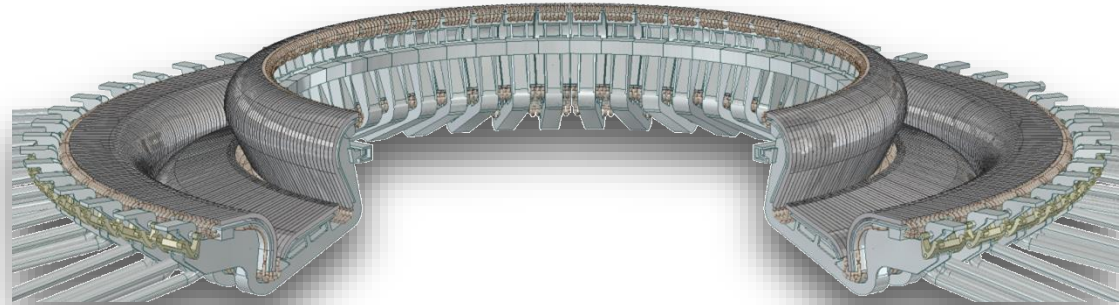
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*More details*

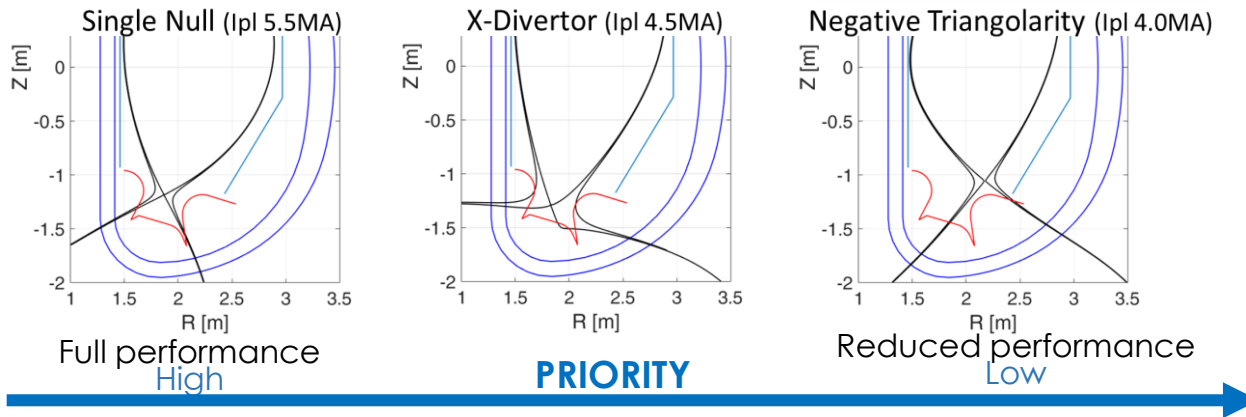
*PL 9 (on Thursday): G. M. Polli, The role of the DTT facility for the development of high heat flux tokamak components,*

*and*

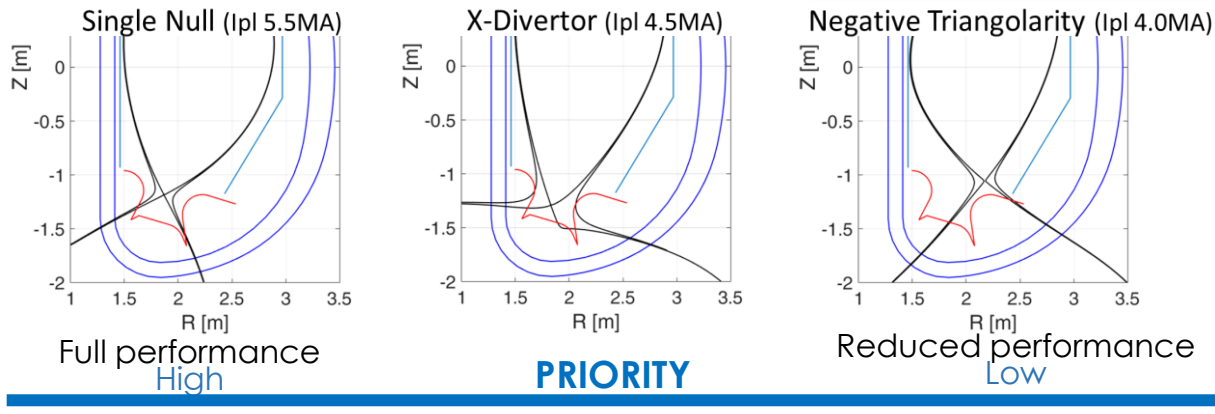
*see PS1-21(on Monday): Marco Utili, Overview of the DTT Assembly Plan*

# First DTT Divertor: conceptual design

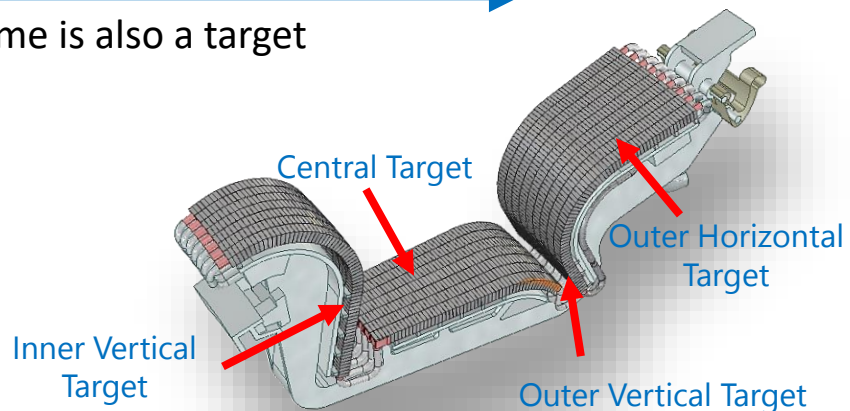
- Compatibility with different magnetic configurations**



- Compatibility with different magnetic configurations

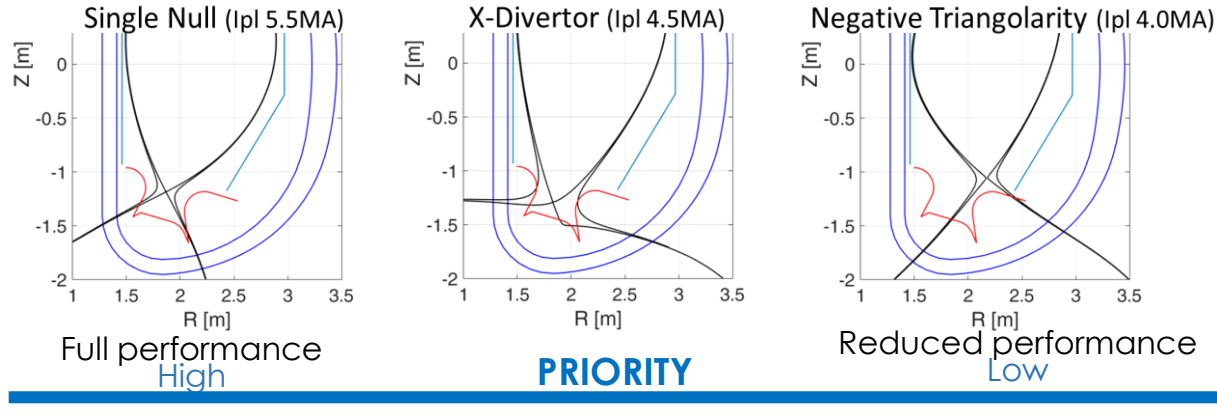


- 4 targets in the DTT Divertor: the straight part of the Dome is also a target

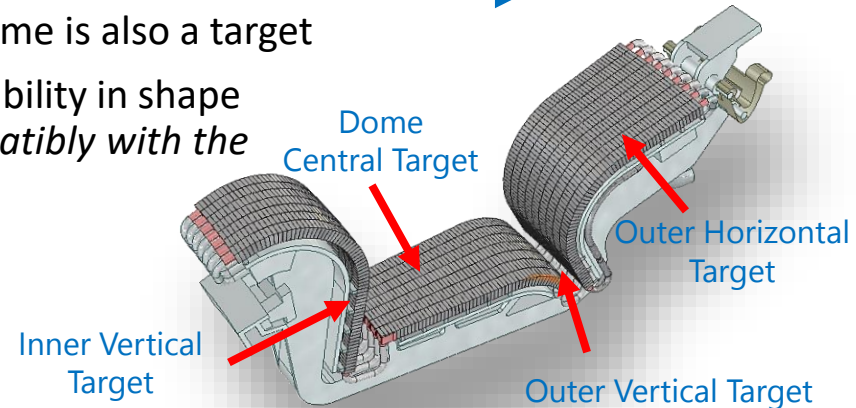




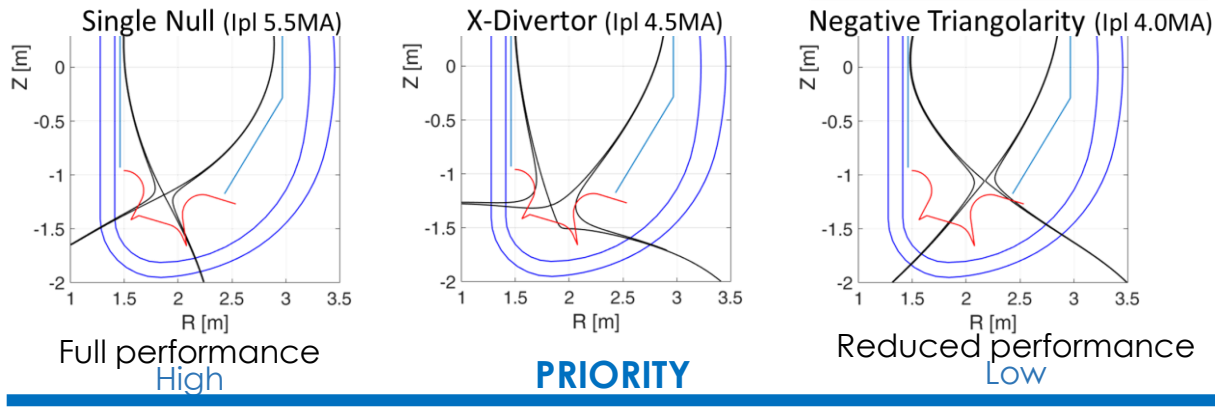
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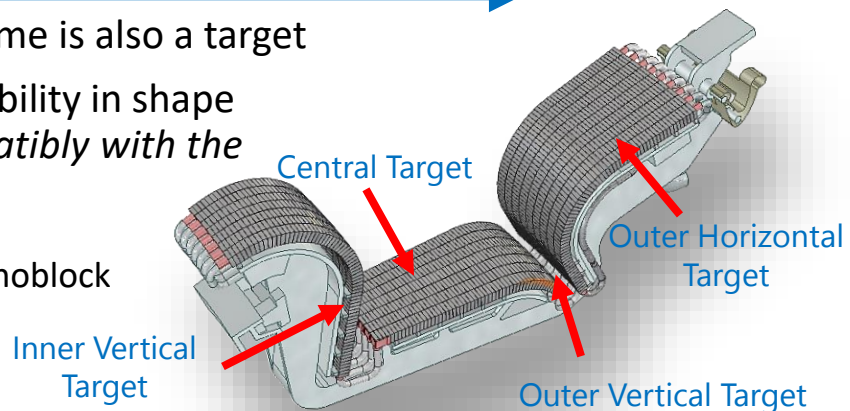
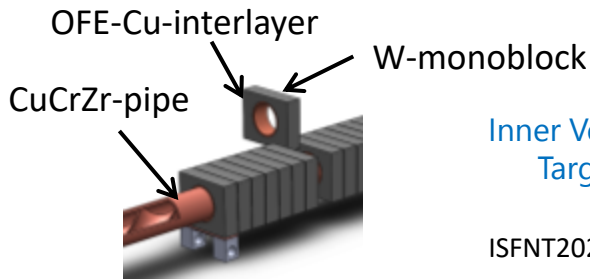
- 4 targets** in the DTT Divertor: the straight part of the Dome is also a target
- The Flat Dome **leave room to the private region** for flexibility in shape and in leg length of the alternative configurations (*compatibly with the space reserved to the cassette body*)



- Compatibility with different magnetic configurations**

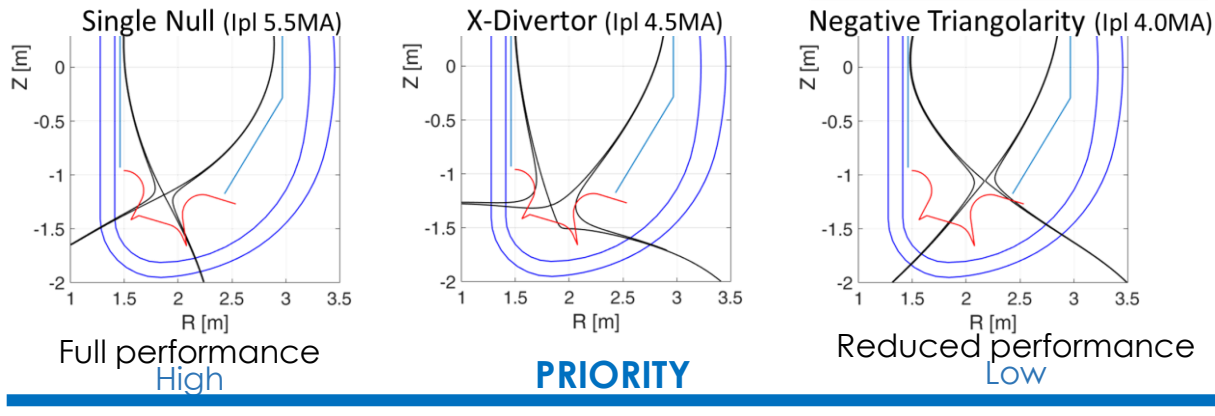


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- the **entire plasma facing surface is in W monoblock**





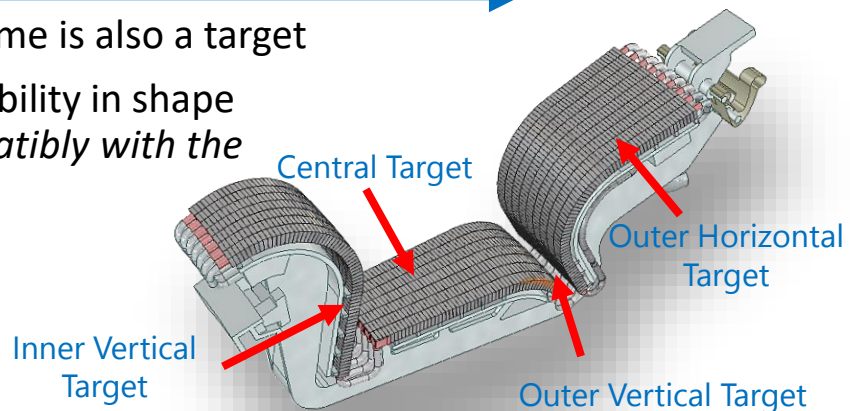
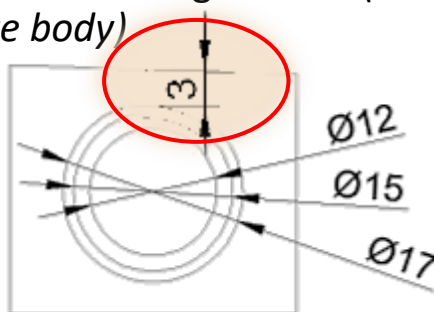
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- 4 targets** in the DTT Divertor: the straight part of the Dome is also a target

- The Flat Dome **leave room to the private region** for flexibility in shape and in leg length of the alternative configurations (*compatibly with the space reserved to the cassette body*)

- the **entire plasma facing surface is in W monoblock** with **reduced armour thickness** to maximize the allowable thermal loads

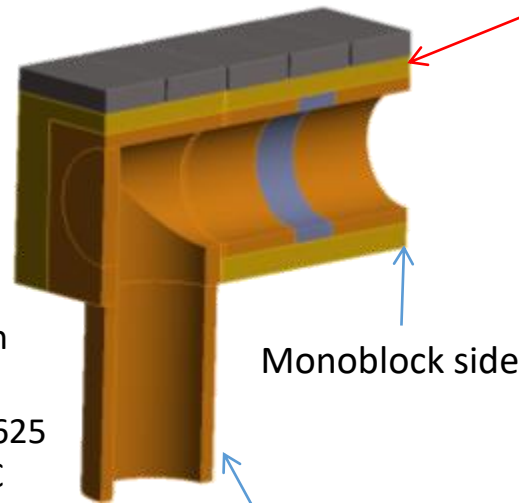
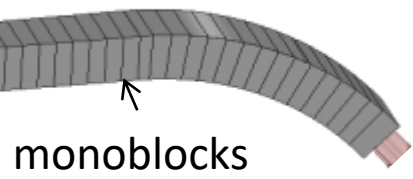






# Conceptual design: poloidal shaping

The Flat Dome requires a 90° bend at the straight end

Flat tiles solution to shield the 90° curve is under study

Flat tiles  
Designed for 5MW/m<sup>2</sup>



-  Tungsten
-  CuCrZr
-  Inconel 625
-  Cu OFHC

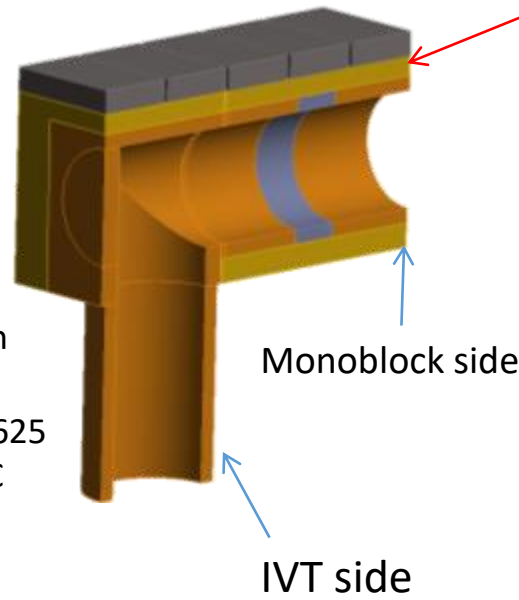
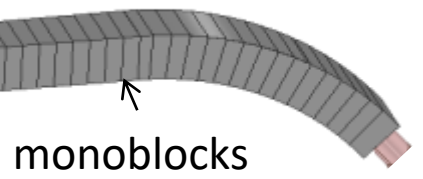
IVT side

# Conceptual design: poloidal shaping

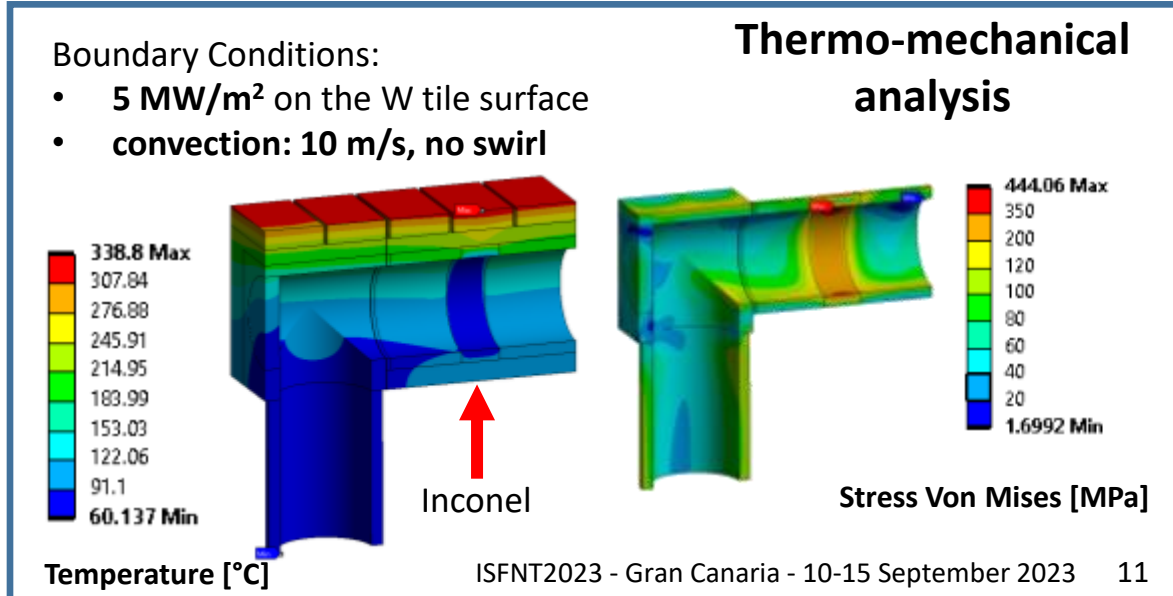
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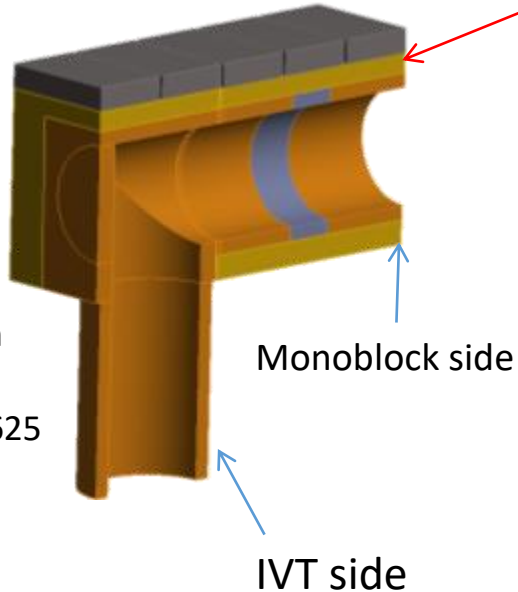
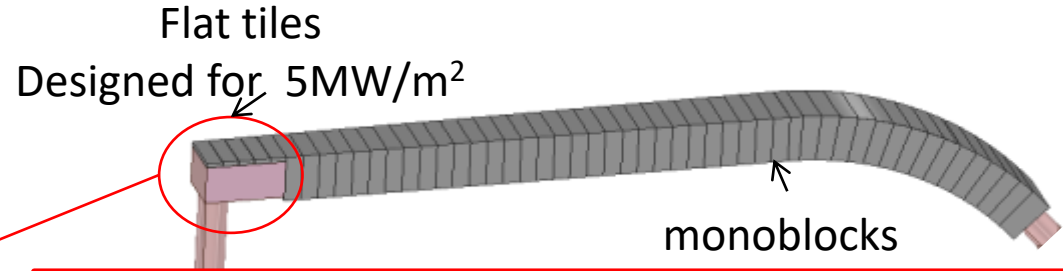
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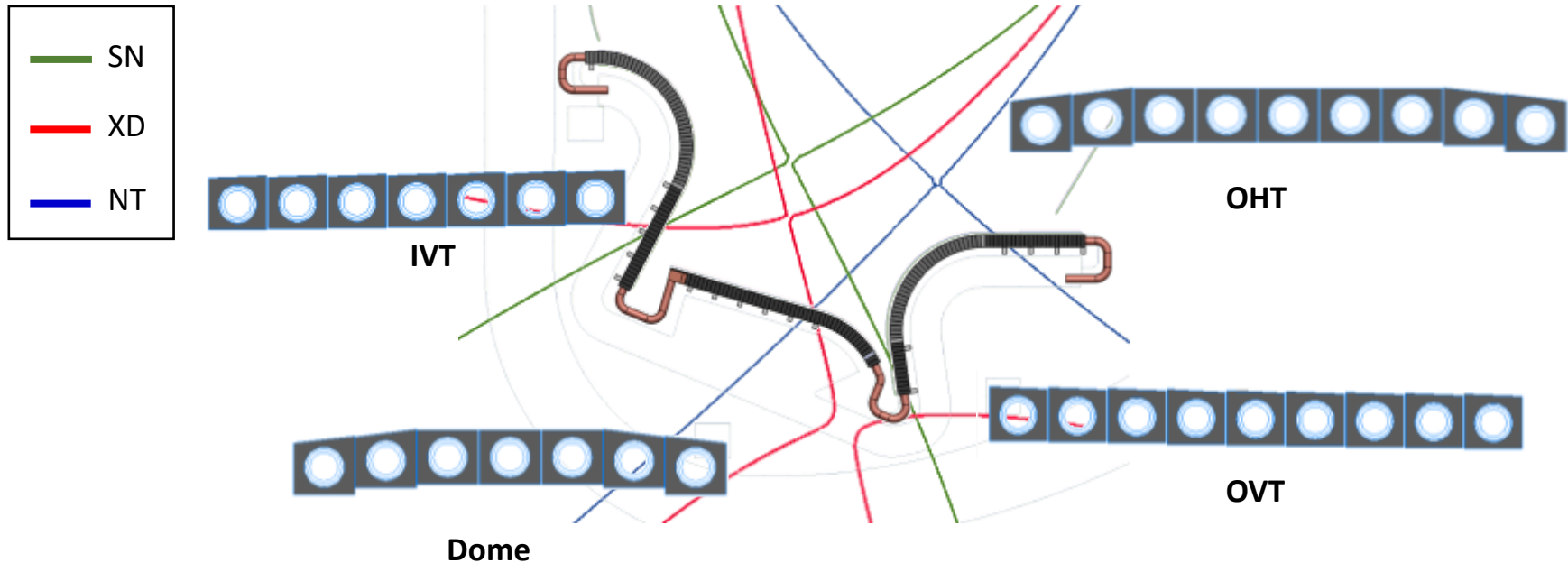
**ISSUES RELATING TO THE 90° BEND**

- The manufacturing process have to be **defined and qualified experimentally**
- The **possibility to insert the Twisted Tape** into the central target **has to be evaluated** because it has to be inserted after the PFU manufacturing (and then after bending the tube)  
↓
- So the **impact of the Critical Heat Flux** on the allowable loads on the central target **without Twisted Tape** must be evaluated.

## Requirements

**IVT e OVT toroidal shaping** : to **maximize the allowable power** on the targets → UNIDIRECTIONAL

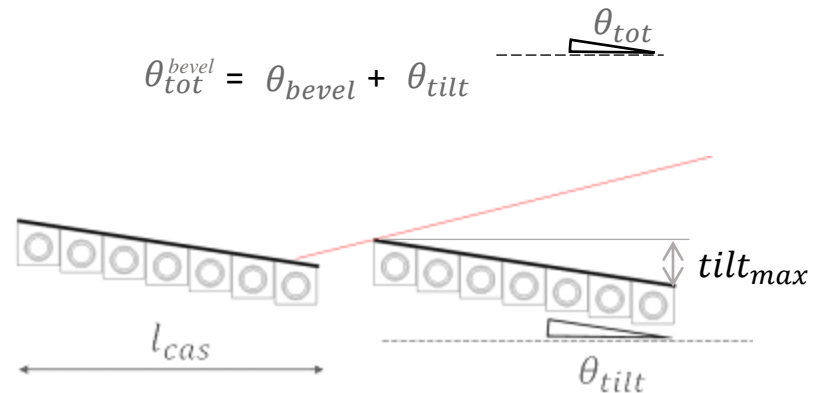
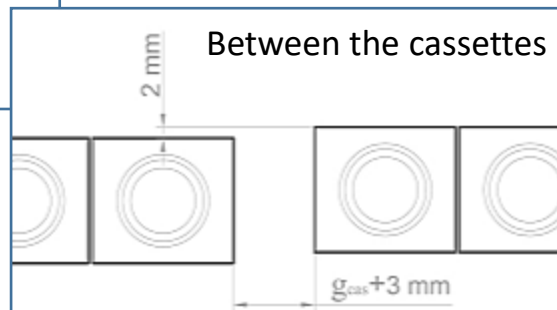
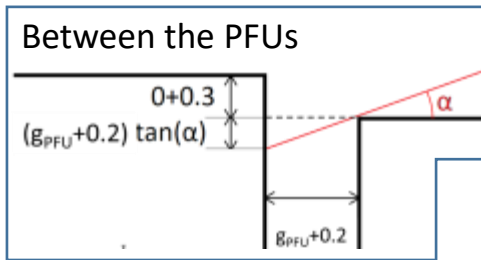
**Dome e OHT toroidal shaping**: for **higher flexibility** → SIMMETRICAL



## Toroidal shaping

	$\alpha$ [°]	$l_{mb}$ [mm]	$g_{casl}$ [mm] (Toll $\pm 3$ mm)	$Step_{casl}$ [mm] (Toll $\pm 2$ mm)	$g_{PFUL}$ [mm] (Toll $\pm 0.2$ mm)	$Step_{PFUL}$ [mm] (Toll $\pm 0.3$ mm)	$\theta_{tot}^{bevel}$ [°]	$tilt_{max}$ [mm]
<b>IVT</b>	2 (SN)	24	8	0	0.4 $\div$ 0.5	0	1.65	1.1
<b>OVT</b>	2 (SN)	24.5	10	0	0.4 $\div$ 0.5	0	1.45	1.0
<b>Dome</b>	5.2 (NT)	25.5	9.5	0	0.4 $\div$ 0.5	0	8.6	6.0
<b>OHT</b>	1.3 (NT)	27.5	12	0	0.4 $\div$ 0.5	0	4.6	2.7

**XD equilibrium does not constrain the toroidal shaping (high flux expansion + shallow grazing angle)**





# Conceptual design: toroidal shaping

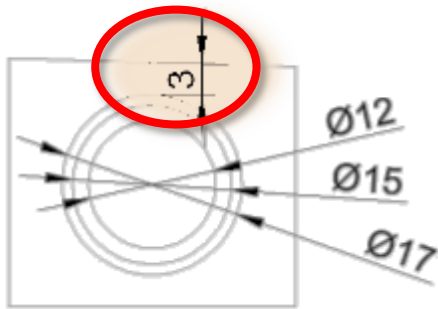
SS and slow transient limit loads

## Inner Vertical Target



$$\theta_{tot}^{bevel} = 1.65^\circ$$

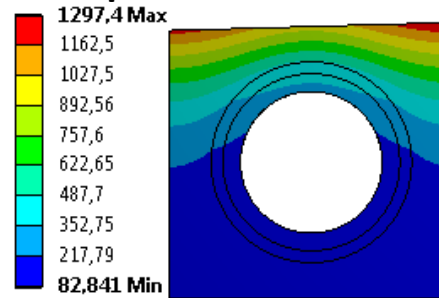
3 mm of armor is compatible with the estimated erosion rate



$h_{tc}$  at 5MPa, 60 C°, 11m/s

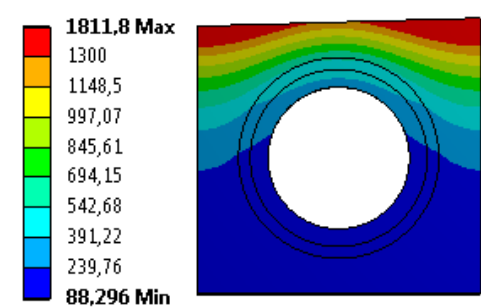
whole block **below** the recrystallization temp.

$q=17.2$  MW/m<sup>2</sup>

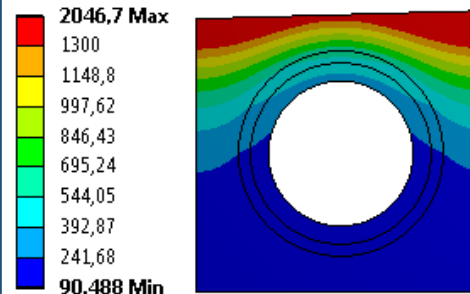


**Central part** below the recrystallization temp.

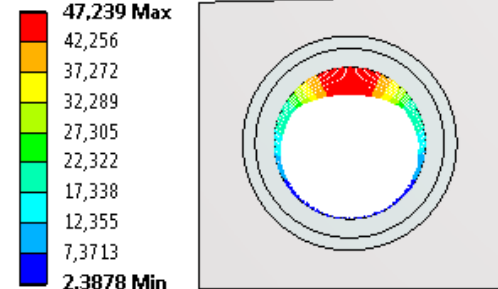
$q=24$  MW/m<sup>2</sup>



**1.4 margin from CHF**



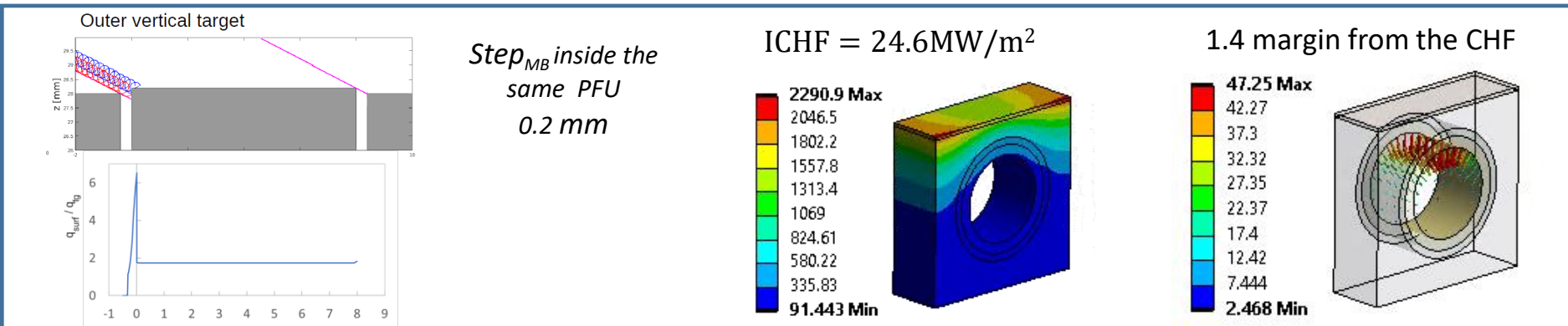
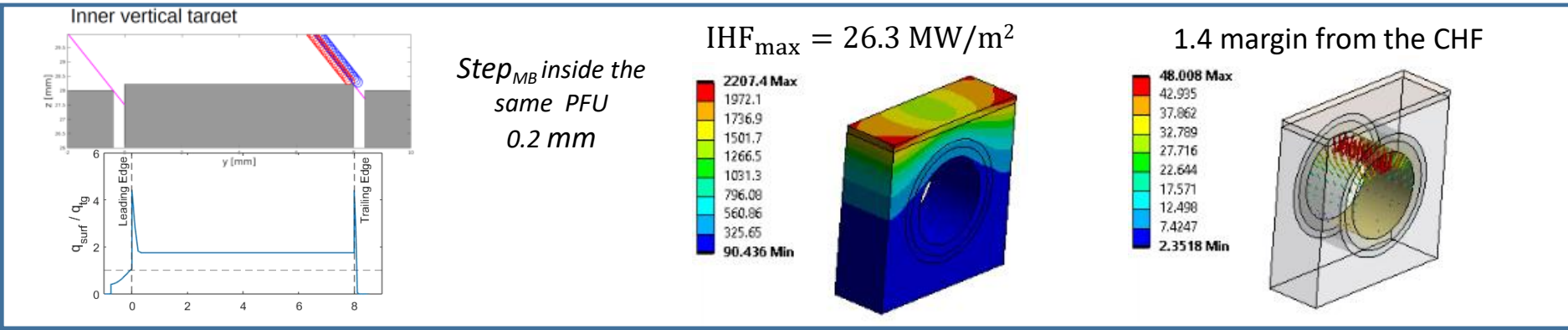
$q=27.2$  MW/m<sup>2</sup>



# Conceptual design: toroidal shaping

## SS and slow transient limit loads: ion orbit modelling

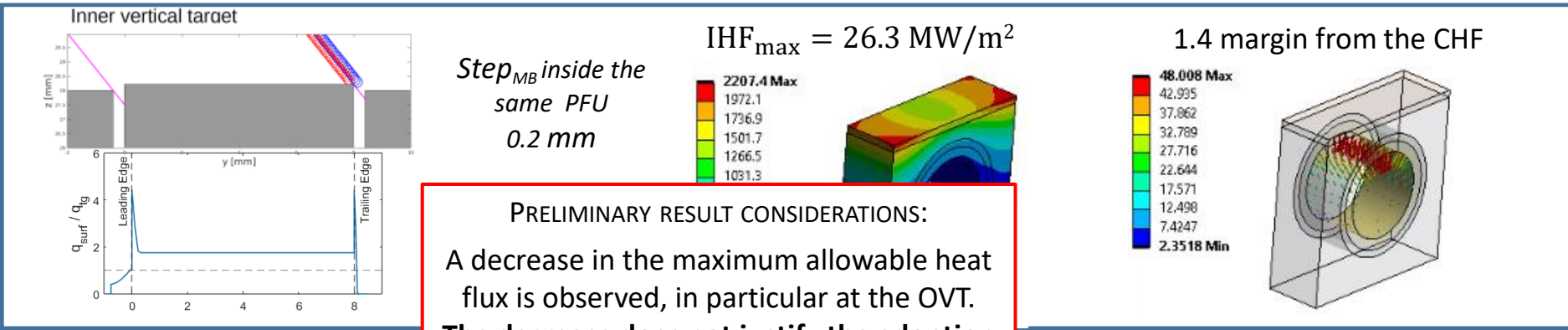
### Gyrokinetic effects on the toroidal leading edges in SN configuration



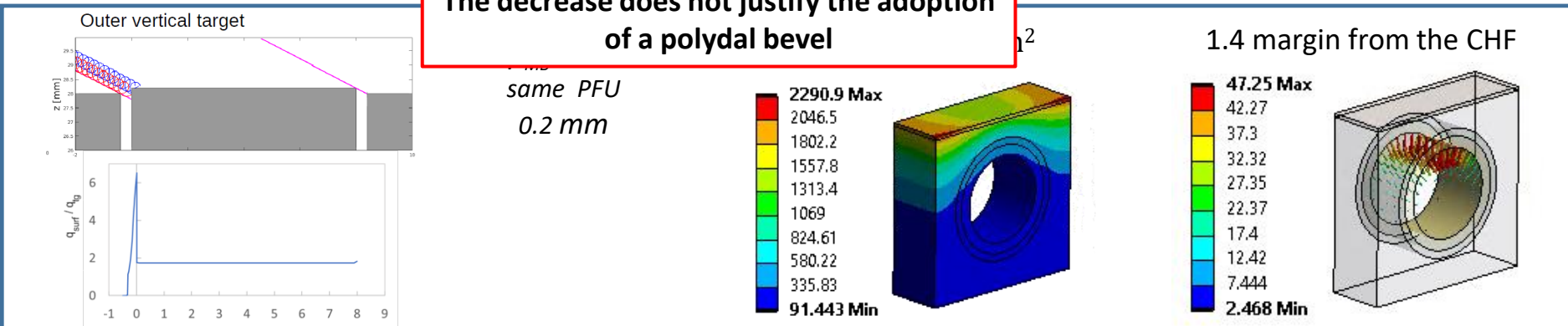
# Conceptual design: toroidal shaping

## SS and slow transient limit loads: ion orbit modelling

### Gyrokinetic effects on the toroidal leading edges in SN configuration



**PRELIMINARY RESULT CONSIDERATIONS:**  
A decrease in the maximum allowable heat flux is observed, in particular at the OVT.  
**The decrease does not justify the adoption of a polydal bevel**



# First DTT Divertor: qualification activities

## Armour thickness assessment

By reducing the thickness of the armor MB plastic deformation is limited and mechanical and physical characteristics of the plasma-facing surface preserved.

The fatigue life must be assessed

## Margin from the Critical Heat Flux evaluation

In particular to evaluate how a Dome without twisted tape is penalizing

## Feasibility study of Flat Tiles design for a 90 ° curve

Small mock-ups have to be manufactured to be tested at thermal fatigue up to 10 MW/m<sup>2</sup>

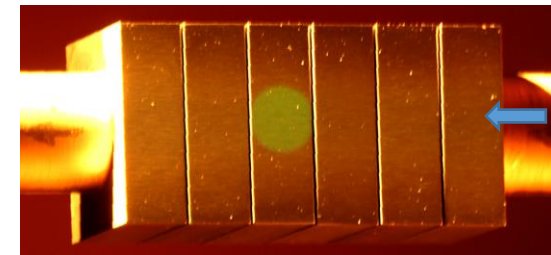
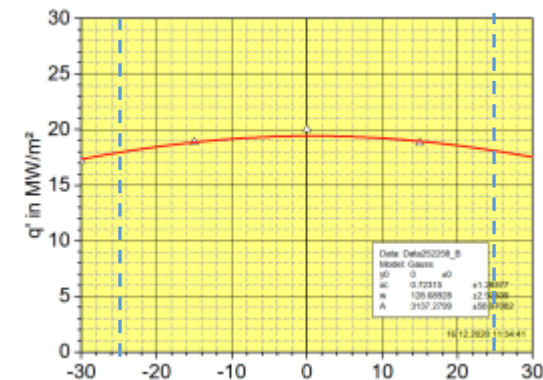
**two mock-ups** were manufactured by HRP in ENEA lab. to be tested at **20 MW/m<sup>2</sup>** per **1000 cycles** in the GLADIS Facility



DTT-T02 mock-up with 4 mm di armour thickness

DTT-T03 mock-up with 3 mm di armour thickness

Applied 20 MW/m<sup>2</sup> heat flux profile



Focus of the two-colour pyrometer

## Technical capabilities of GLADIS

- Power 2 x 1 MW ion source, H<sub>2</sub>
- U<sub>ex</sub> 15 – 55 kV
- Heat flux 3 - 45 MW/m<sup>2</sup>, 150 mm FWHM  
∅ 50 mm (95% q' max)
- Pulse length 1 ms - 45 s
- Cycle rate 80 - 100 /h
- Target dim. mm size up to 60 cm (max. 2m)
- Target cooling: 20-230 °C inlet, 1-4 MPa, < 8l/s  
„high purity water“ fulfils ITER requirements
- Capability for operation with H/He beam



## hydraulic conditions during the test

T<sub>in</sub> = 130°C;

p<sub>in</sub> 40 bar;

v = 14 m/s;

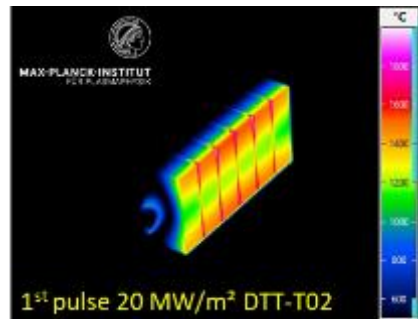
← p<sub>in</sub> 50 bar;

← v 11m/s

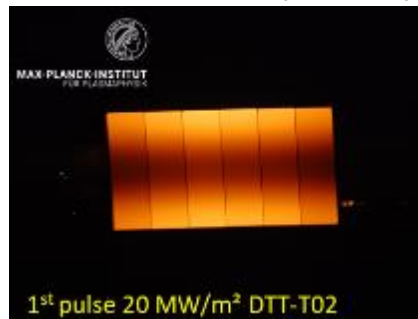
lower available pressure -> increased the flow rate to verify the margin 1.4 from the critical heat flux

## Ongoing HHF tests

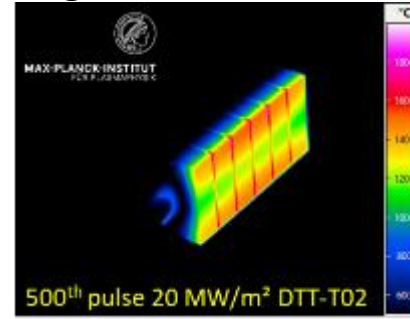
### DTT-T02 (4 mm) during the HHF test



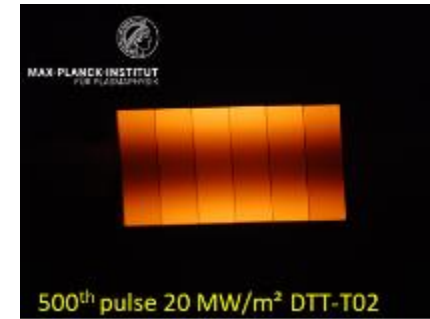
infrared camera



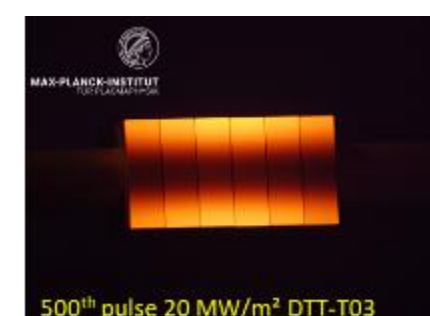
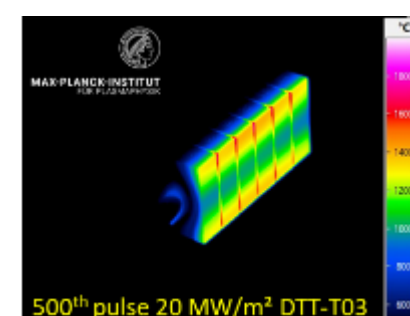
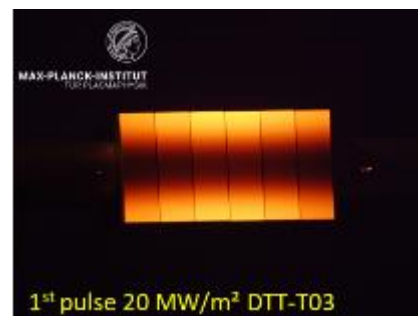
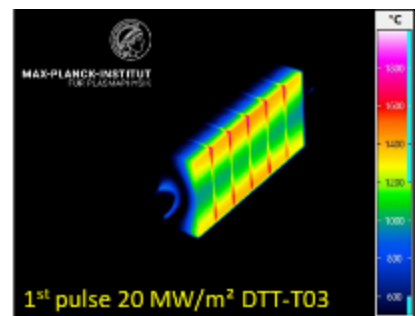
CCD camera



infrared camera



CCD camera



### DTT-T03 (3 mm) during the HHF test



## Ongoing HHF tests

DTT-T02 (4 mm) during the HHF test



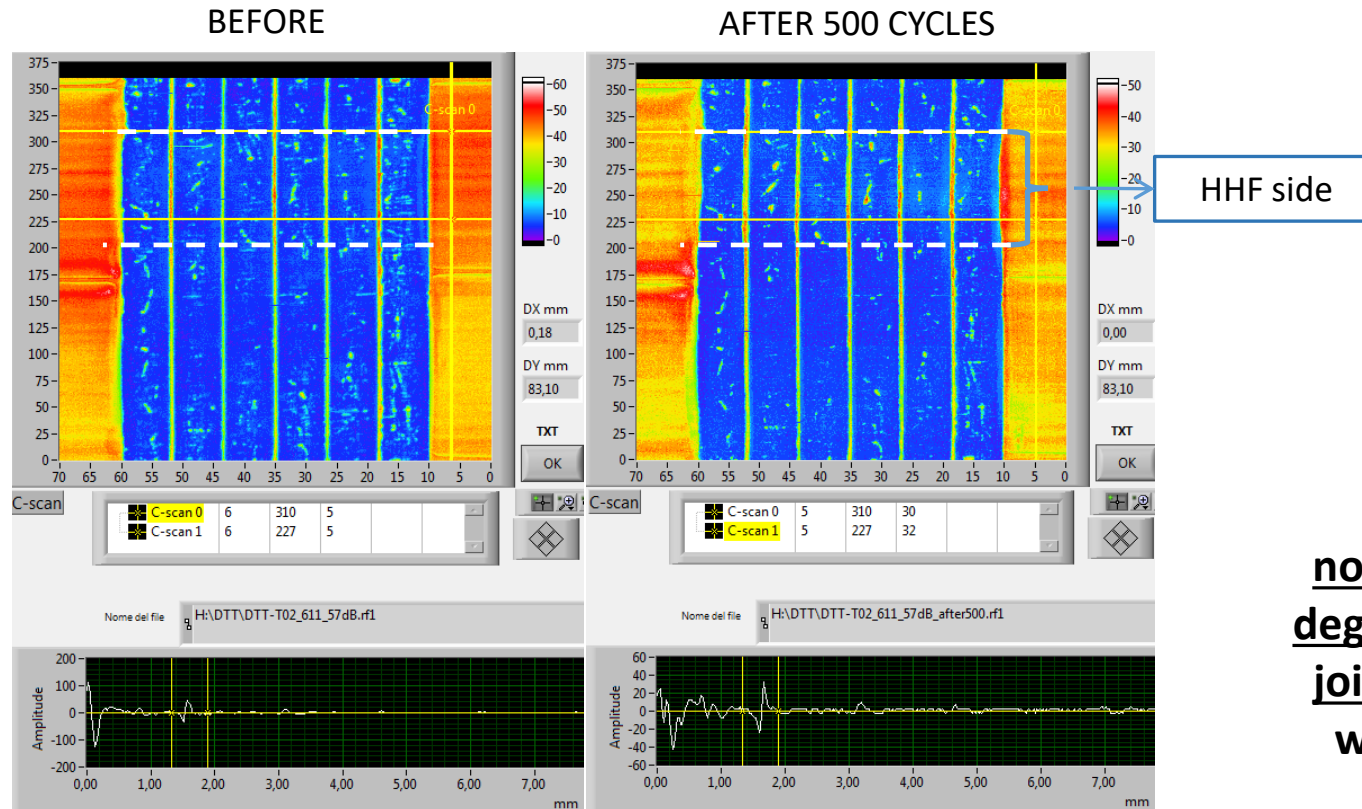
➔ No indication of any defects and no change of temperature distribution during cycling on both mock-ups.



The remaining 500 cycles will be performed in October

DTT-T03 (3 mm) during the HHF test

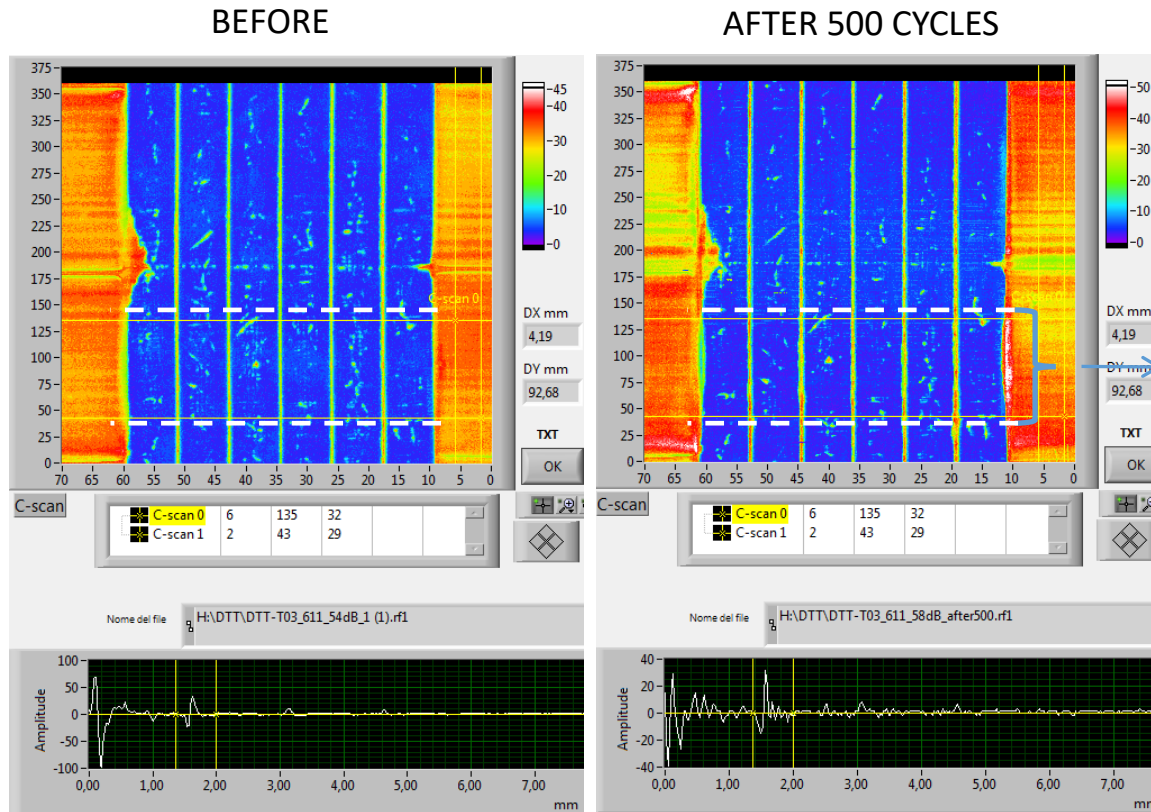
## Intermediate results: UT results after 500 x 20 MW/m<sup>2</sup> - DTT-T-02



C-scan at HRP interface

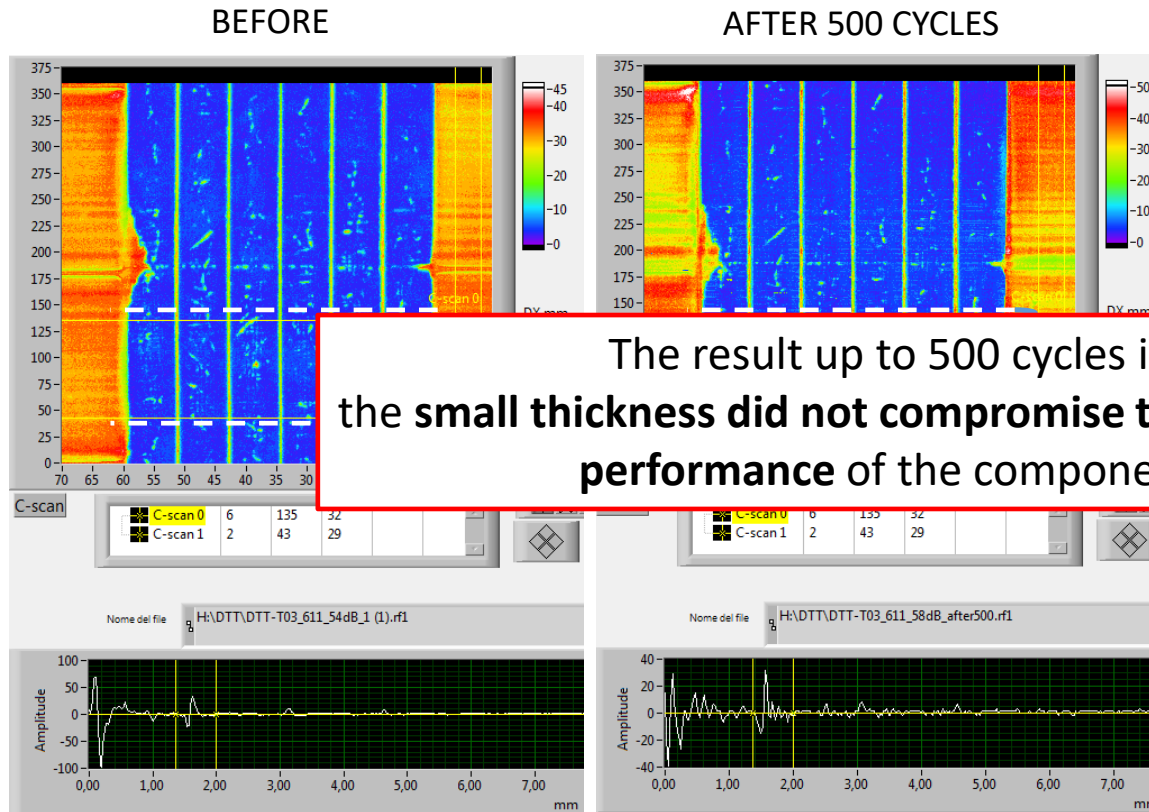
**no indications of degradation of the jointed interface were detected**

## Intermediate results: UT results after 500 x 20 MW/m<sup>2</sup> - DTT-T-03



C-scan at HRP interface

## Intermediate results: UT results after 500 x 20 MW/m<sup>2</sup> - DTT-T-03



The result up to 500 cycles is  
the **small thickness did not compromise the mechanical performance of the component**

**no indications of degradation of the jointed interface were detected**

C-scan at HRP interface

## Motivation for the experimental campaign

standard operating conditions for the first divertor



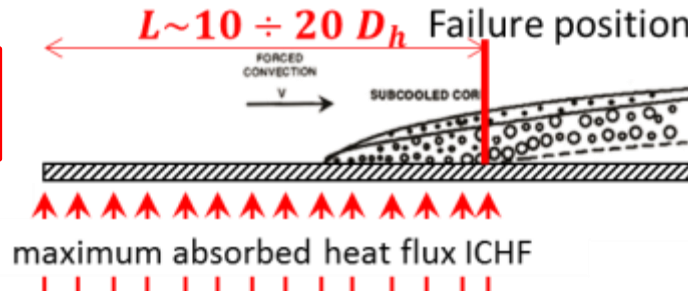
	WCHF* [MW/m <sup>2</sup> ]		WHF <sub>max</sub> = WCHF/1.4 [MW/m <sup>2</sup> ]		IHF <sub>max</sub> ** [MW/m <sup>2</sup> ]	
	IVT/OVT Swirl	Dome No Swirl	IVT/OVT Swirl	Dome No Swirl	IVT/OVT Swirl	Dome No Swirl
T=30 °C	86.98	58.8	62.1	41.9	35.5	23.5
T=60 °C	66.3	45.2	47.3	32.3	27.3	18.46

\*by TONG<sub>Dh</sub>-CEA correlation

\*\*by ANSYS simulations

Swirl: Twisted Tape, Y=pitch/2Dh=2, Thickness=0.8 mm, v=11 m/s, P=5MPa

The CHF depends on:  
**Loaded axial length**



the semi-empirical correlations estimate the CHF in case the loaded length is between 10 and 20 times the hydraulic diameter

## Motivation for the experimental campaign

standard operating conditions for the first divertor

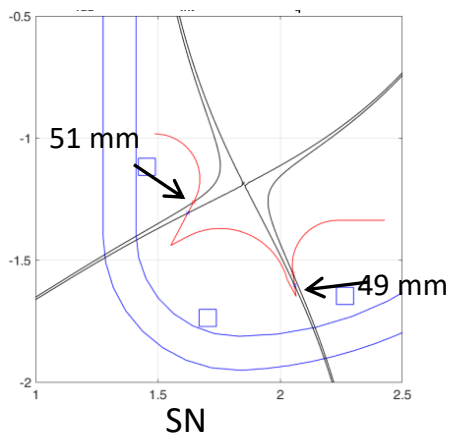


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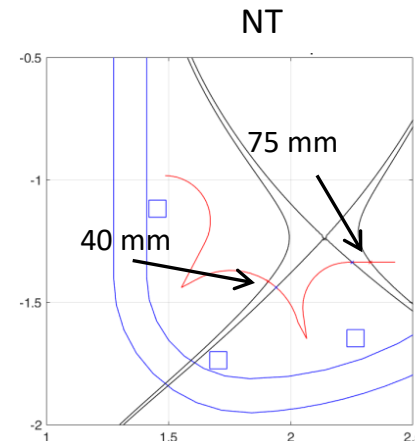
\*\*by ANSYS simulations

Swirl: Twisted Tape, Y=pitch/2Dh=2, Thickness=0.8 mm, v=11 m/s P=5MPa



The DTT PFU have the same hydraulic diameter as in ITER and DEMO, but a **much smaller loaded area**

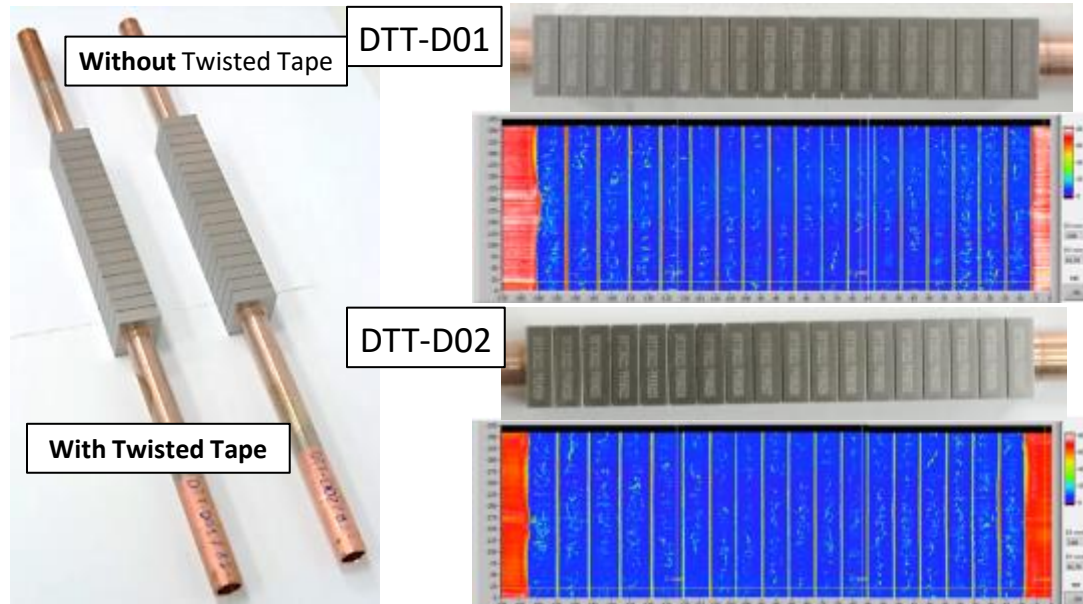
1.4 of Margin may be too conservative when applied to DTT





## Purpose of the tests

**Two 18 monoblock mock-ups** were manufactured by HRP to be tested at the HADES Facility (electron-beam gun).  
Two more identical mock-ups will be manufactured and tested later this year

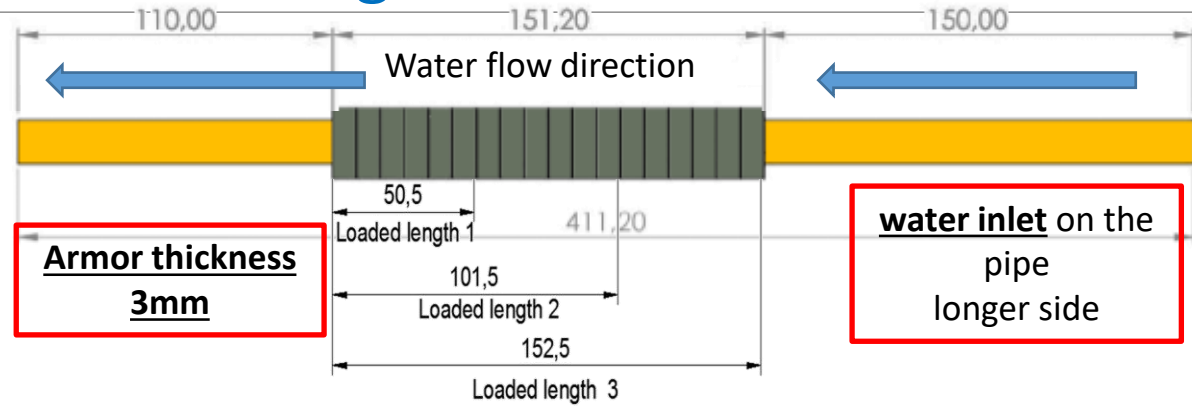


### Purposes of the tests are:

- **Assess the actual CHF at DTT standard hydraulic conditions (60°C) for a loaded axial length comparable with the plasma footprint (SN and NT scenarios)**
- **quantify the relationship between the loaded length and CHF**

The UT examination show the good manufacturing of the mock-ups

## Test matrix



### Max available load in HADES

~13 [MW/m<sup>2</sup>]@Loaded length 3

~19[MW/m<sup>2</sup>]@ Loaded length 2

~35 [MW/m<sup>2</sup>]@Loaded length 1

v=11 m/s; P=3.5 bar

### Without Twisted Tape (D02)

$T_w$ [°C]	$fp^{**}$	IHF [MW/m <sup>2</sup> ]					$T_{max}$ [°C] @35MW/m <sup>2</sup>
		CHF*	ICHF**	Loaded length 3	Loaded length 2	Loaded length 1	
60	1.78	39.2	22	-	-	from 15-to 35 or CHF	3234
140	1.8	21.4	11.8	from 5 - to CHF	from 7- to CHF	from 8- to CHF	

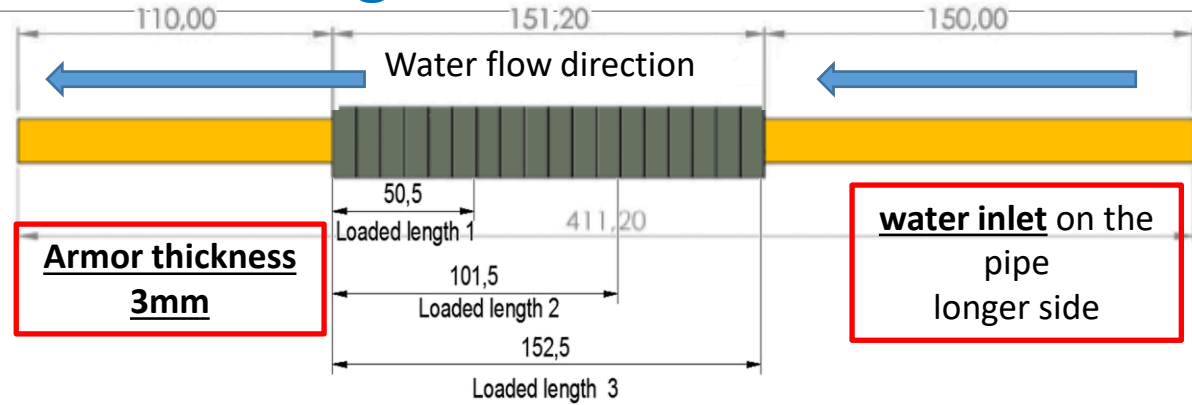
### With Twisted Tape (D01)

$T_w$ [°C]	$fp^{**}$	IHF [MW/m <sup>2</sup> ]					$T_{max}$ [°C] @35MW/m <sup>2</sup>
		CHF*	ICHF**	Loaded length 3	Loaded length 2	Loaded length 1	
60	1.85	59.1	32	-	-	from 10- to 35 or CHF	3152
200	1.8	20.9	11.6	from 5 - to CHF	from 5 - to CHF	from 9- toCHF	

\*by  $TONG_{Dh}$ -CEA correlation

\*\*ANSYS simulations

## Test matrix



**Max available load in HADES**  
 ~13 [MW/m<sup>2</sup>]@Loaded length 3  
 ~19[MW/m<sup>2</sup>]@ Loaded length 2  
 ~35 [MW/m<sup>2</sup>]@Loaded length 1

v=11 m/s; P=3.5 bar

Without Twisted Tape (D02)							
$T_w$ [°C]	$fp^{**}$	IHF [MW/m <sup>2</sup> ]					$T_{max}$ [°C] @35MW/m <sup>2</sup>
		CHF*	ICHF**	Loaded length 3	Loaded length 2	Loaded length 1	
60	1.78	39.2	22	-	-	from 15-to 35 or CHF	3234
140	1.8	21.4	11.8	from 5 - to CHF	from 7- to CHF	from 8- to CHF	

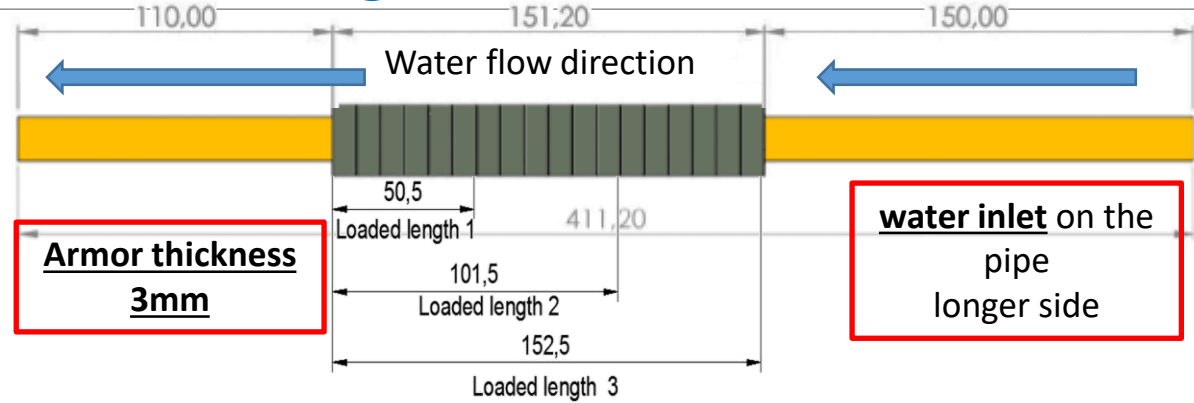
Assess the actual CHF the load is applied over a **length of 50 mm** (close to the footprint) and a water temperature of 60°C

$T_w$ [°C]	$fp^{**}$	IHF [MW/m <sup>2</sup> ]					$T_{max}$ [°C] @35MW/m <sup>2</sup>
		CHF*	ICHF**	Loaded length 3	Loaded length 2	Loaded length 1	
60	1.85	59.1	32	-	-	from 10- to 35 or CHF	3152
200	1.8	20.9	11.6	from 5 - to CHF	from 5 - to CHF	from 9- toCHF	

\*by  $TONG_{Dh}$ -CEA correlation

\*\*ANSYS simulations

## Test matrix



### Max available load in HADES

~13 [MW/m<sup>2</sup>]@Loaded length 3

~19[MW/m<sup>2</sup>]@ Loaded length 2

~35 [MW/m<sup>2</sup>]@Loaded length 1

$v=11$  m/s;  $P=3.5$  bar

Without Twisted Tape (D02)							
$T_w$ [°C]	$fp^{**}$	IHF [MW/m <sup>2</sup> ]					$T_{max}$ [°C] @35MW/m <sup>2</sup>
		CHF*	ICHF**	Loaded length 3	Loaded length 2	Loaded length 1	
60	1.78	39.2	22	-	-	From 15-to 35 or CHF	3234
140	1.8	21.4	11.8	from 5 - to CHF	from 7- to CHF	from 8- to CHF	

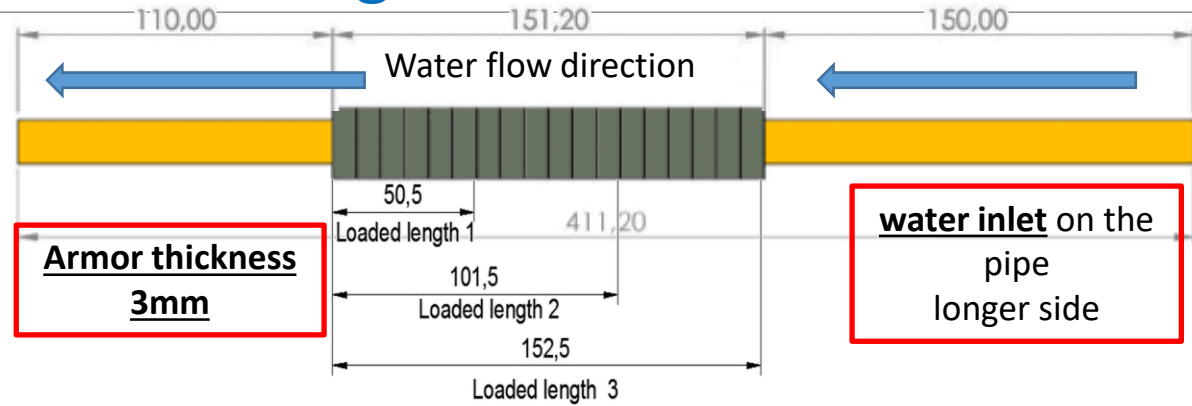
To quantify the relationship between the loaded length and CHF, three axial lengths are loaded. The water temperature was increased so that the CHF could also be achieved loading the length 3

$T_w$ [°C]	$fp^{**}$	IHF [MW/m <sup>2</sup> ]					$T_{max}$ [°C] @35MW/m <sup>2</sup>
		CHF*	ICHF**	Loaded length 3	Loaded length 2	Loaded length 1	
60	1.85	59.1	32	-	-	from 10- to 35 or CHF	3152
200	1.8	20.9	11.6	from 5 - to CHF	from 5 - to CHF	from 9- toCHF	

\*by  $TONG_{Dh}$ -CEA correlation

\*\*ANSYS simulations

## Test matrix



### Max available load in HADES

~13 [MW/m<sup>2</sup>]@Loaded length 3

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~35 [MW/m<sup>2</sup>]@Loaded length 1

v=11 m/s; P=3.5 bar

Without Twisted Tape (D02)							
$T_w$ [°C]	$fp^{**}$	IHF [MW/m <sup>2</sup> ]					$T_{max}$ [°C] @35MW/m <sup>2</sup>
		CHF*	ICHF**	Loaded length 3	Loaded length 2	Loaded length 1	
60	1.78	39.2	22	-	-	From 15-to 35 or CHF	3234
140	1.8	21.4	11.8	from 5 - to CHF ✓	from 7- to CHF ✓	from 8- to CHF	

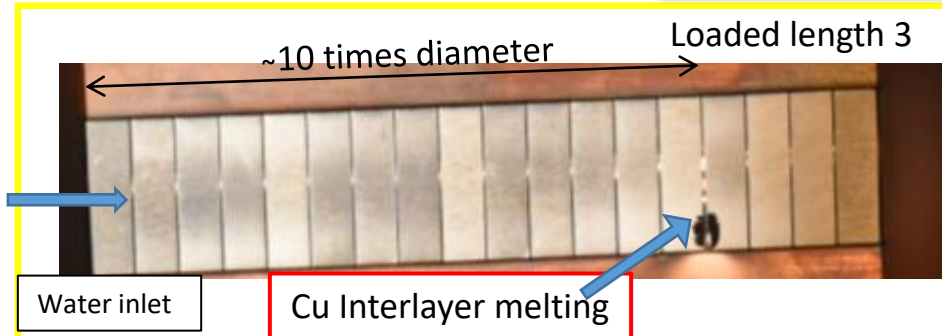
With Twisted Tape (D01)							
$T_w$ [°C]	$fp^{**}$	IHF [MW/m <sup>2</sup> ]					$T_{max}$ [°C] @35MW/m <sup>2</sup>
		CHF*	ICHF**	Loaded length 3	Loaded length 2	Loaded length 1	
60	1.85	59.1	32	-	-	10-35 or CHF ✓	3152
200	1.8	20.9	11.6	from 5 - to CHF ✓	from 5 - to CHF ✓	from 9- toCHF	

\*by  $TONG_{Dh}$ -CEA correlation

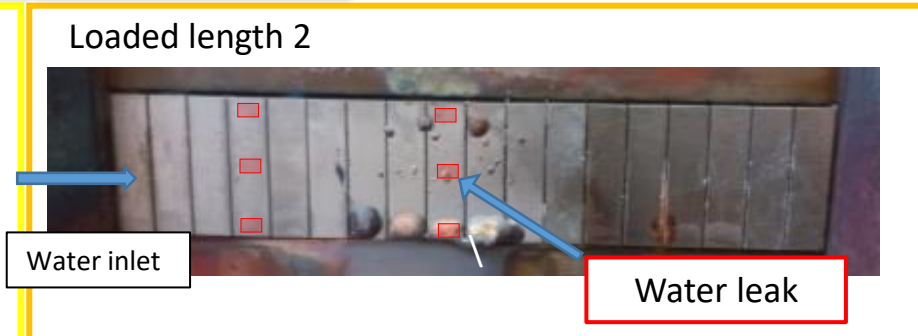
\*\*ANSYS simulations

## Preliminary results: without twisted tape

Without Twisted Tape (D02)							
$T_w$ [°C]	$fp^{**}$	IHF [MW/m <sup>2</sup> ]					$T_{max}$ [°C] @35MW/m <sup>2</sup>
		CHF*	ICHF**	Loaded length 3	Loaded length 2	Loaded length 1	
60	1.78	39.2	22	-	13-19	15-35 or CHF	3234
140	1.8	21.4	11.8	5- CHF	7-CHF	8-CHF	



Excellent agreement with Tong-CEA correlation  
 CHF occurred at a distance of 10 times the diameter at  
 12.3 MW/m<sup>2</sup> (by correlation 11.8 MW/m<sup>2</sup>)



CHF occurred at a distance of 76mm at 14 MW/m<sup>2</sup> (by  
 correlation 11.8 MW/m<sup>2</sup>). CHF increased by 14%  
 reducing loaded length by 1/3

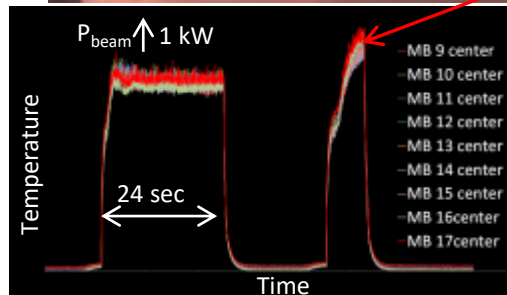
due to the breakage of the mock-up, the test was interrupted  
**A mock-up of 6 monoblocks will be used to finish the tests**



## Preliminary results: with twisted tape

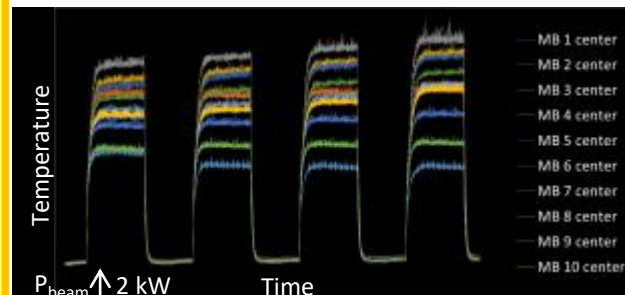
With Twisted Tape (D01)							
$T_w$ [°C]	$fp^{**}$	IHF [MW/m <sup>2</sup> ]					$T_{max}$ [°C] @35MW/m <sup>2</sup>
		CHF*	ICHF**	Loaded length 3	Loaded length 2	Loaded length 1	
60	1.85	59.1	32	-	-	10-35 or CHF	3152
200	1.8	20.9	11.6	5-CHF	5-CHF	9-CHF	

Loaded length 3



sudden temperature increase in the MBs far from the water inlet @ 12.3 MW/m<sup>2</sup>

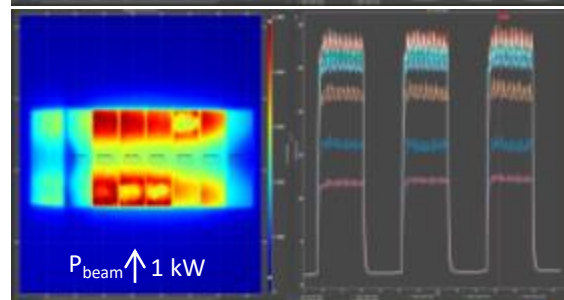
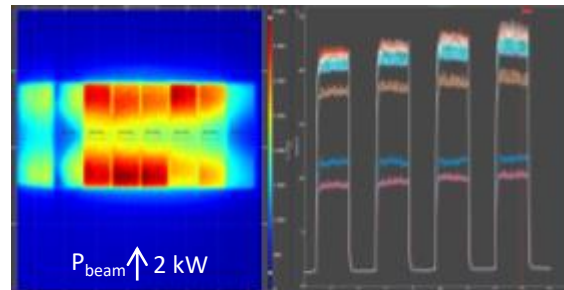
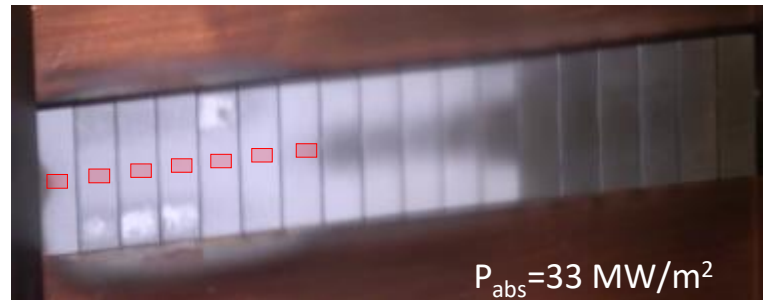
Loaded length 2



No indication until 15.5 MW/m<sup>2</sup>.  
**To be continued**

## Preliminary results: with twisted tape

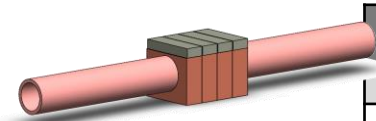
With Twisted Tape (D01)							
$T_w$ [°C]	$fp^{**}$	IHF [MW/m <sup>2</sup> ]					$T_{max}$ [°C] @35MW/m <sup>2</sup>
		CHF*	ICHF**	Loaded length 3	Loaded length 2	Loaded length 1	
60	1.85	59.1	32	-	-	10-35 or CHF	3152
200	1.8	20.9	11.6	5-CHF	5-CHF	9-CHF	



Loaded length 1 (60°C)

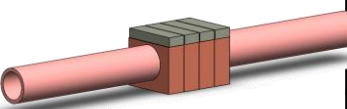
No indication of CHF until the Tungsten melting @33 MW/m<sup>2</sup>

## Progress in the FT mock-up manufacturing

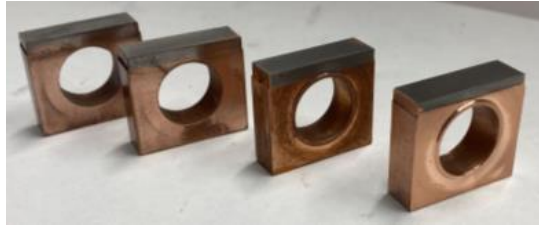


HHF Testing parameters				
Cycles	q [MW/m <sup>2</sup> ]	T <sub>w</sub> [°C]	p [bar]	v [m/s]
5000	5	130	40	11
300	10	130	40	11

## Progress in the FT mock-up manufacturing

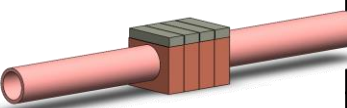


HHF Testing parameters				
Cycles	q [MW/m <sup>2</sup> ]	T <sub>w</sub> [°C]	p [bar]	v [m/s]
5000	5	130	40	11
300	10	130	40	11

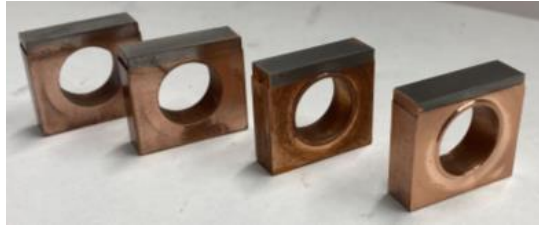


W Flat Tiles on Cu monoblocks  
manufactured by Cu casting  
(bubbles detected by UT at interface  
with W)

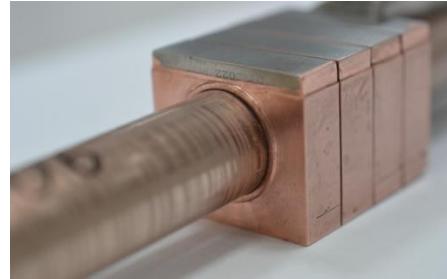
## Progress in the FT mock-up manufacturing



HHF Testing parameters				
Cycles	q [MW/m <sup>2</sup> ]	T <sub>w</sub> [°C]	p [bar]	v [m/s]
5000	5	130	40	11
300	10	130	40	11

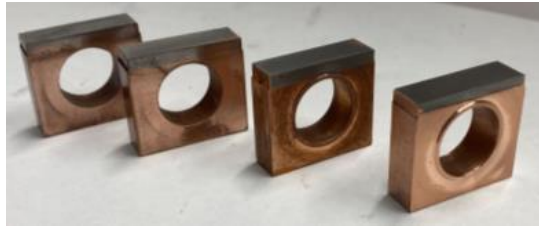
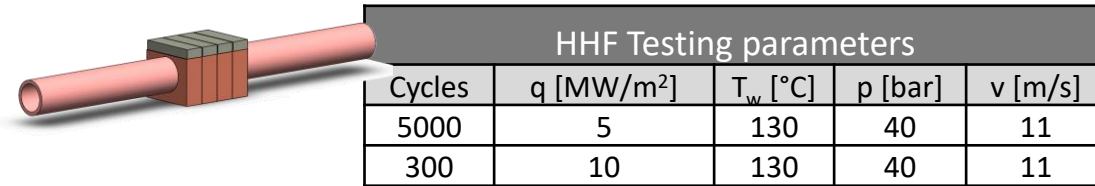


W Flat Tiles on Cu monoblocks  
manufactured by Cu casting  
(bubbles detected by UT at interface  
with W)

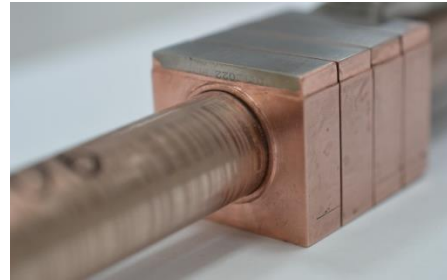


Flat tiles mock-up by HRP

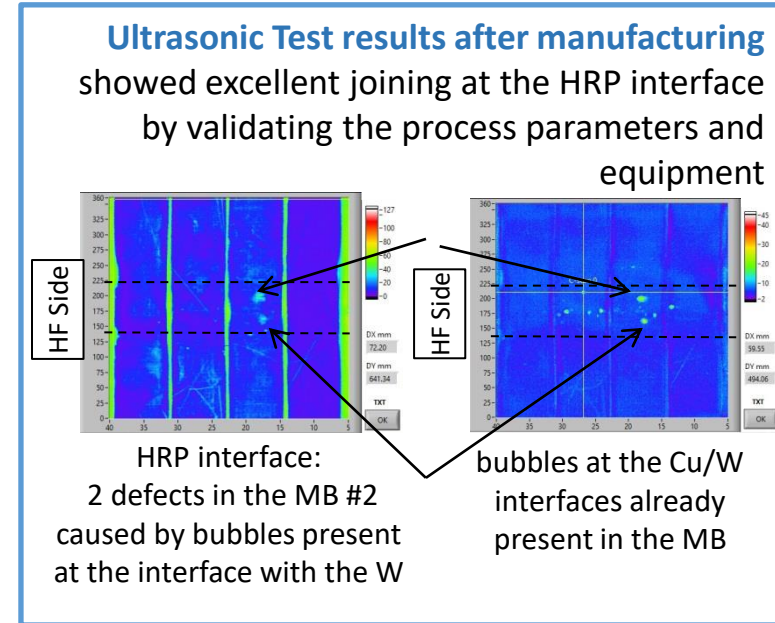
## Progress in the FT mock-up manufacturing



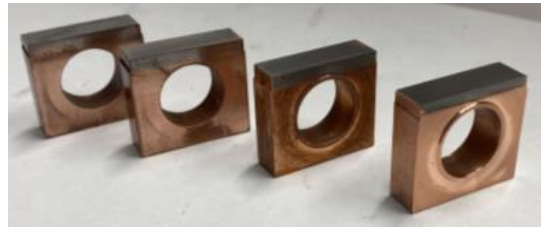
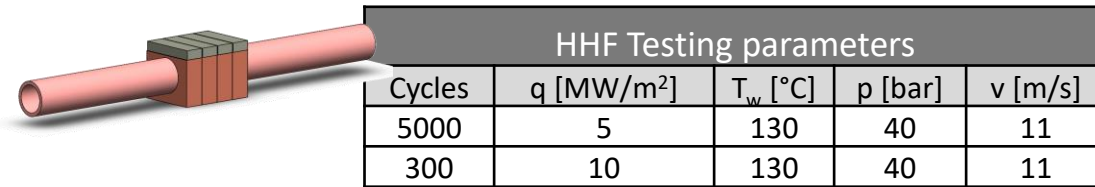
W Flat Tiles on Cu monoblocks manufactured by Cu casting (bubbles detected by UT at interface with W)



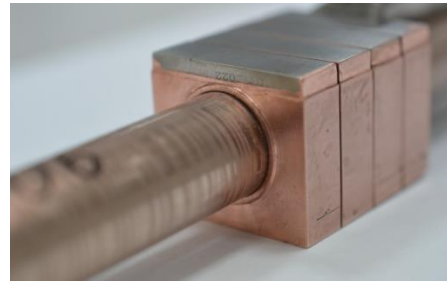
Flat tiles mock-up by HRP



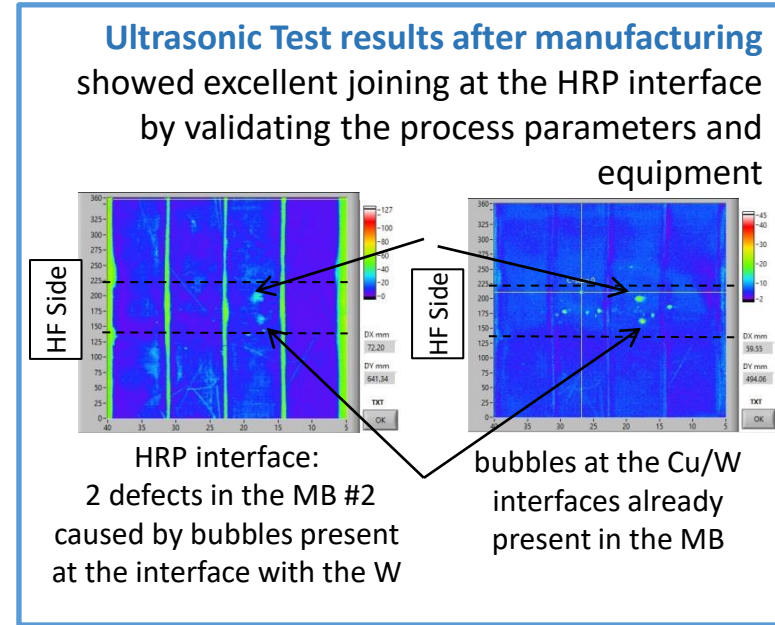
## Progress in the FT mock-up manufacturing



W Flat Tiles on Cu monoblocks manufactured by Cu casting (bubbles detected by UT at interface with W)



Flat tiles mock-up by HRP



The **parameters of the HRP process have been modified to obtain the junction between the Cu block and CuCrZr pipe.**

The **same parameters** were used for the fabrication of **the W monoblock small mock-ups** under testing HADES and GLADIS with positive results.

**FT at the Dome extremity and W monoblocks can be joined to CuCrZr tubes during the same HRP cycle**



- The purpose of this **first** divertor is
  - to be **flexible to test different magnetic configurations and scenarios** thanks to **large private region room** and
  - to be **reliable** thanks to the **high allowable thermal loads**.
- Its design was born taking into account the manufacturing constraints, **speeding up and simplifying the qualification phase**
- The design is in the engineering phase.
- The **qualification** of the components is ongoing with positive **results**.
- **Full scale prototype** planned for late next year.

# Thanks for your attention

and thanks to the  
In-Vessel Component team

## DTT:

*IVC WBS Coor.: B.Riccardi, DIV RO: F. Giorgetti, FW RO: M. Furno Palumbo*

## ENEA Labs:

*M. Angelucci, E. Cacciotti, F. Crea, F. Crescenzi, R. De Luca, G. De Sano (Tor Vergata phd student), G. Dose, P. Frosi, M. Iafrati, P. Lorusso, A. Mancini, E. Martelli, A. Moriani, A. Satriano, A. Tatì, L. Verdini and S. Roccella*

## RFX:

*P. Innocente, L. Balbinot (UniTus phd student)*

## PoliMi:

*M. Passoni*

## CREATE:

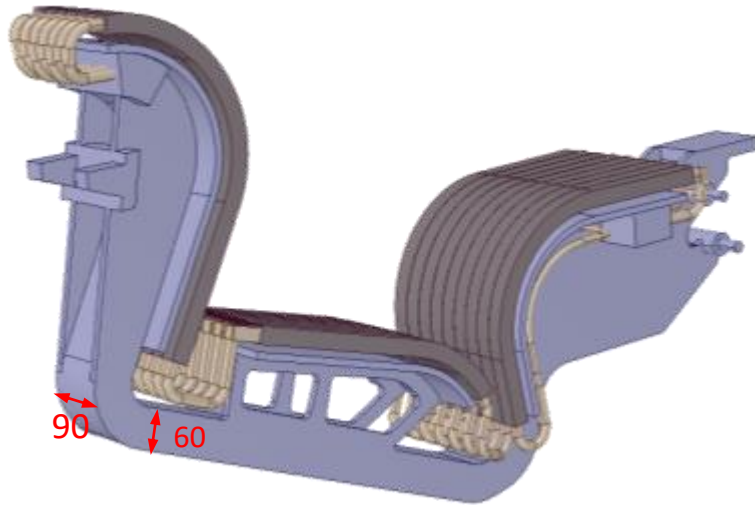
*D. Marzullo, N. Massanova*

## JSI:

*B. Končar , O.Costa and P. Tarfila*

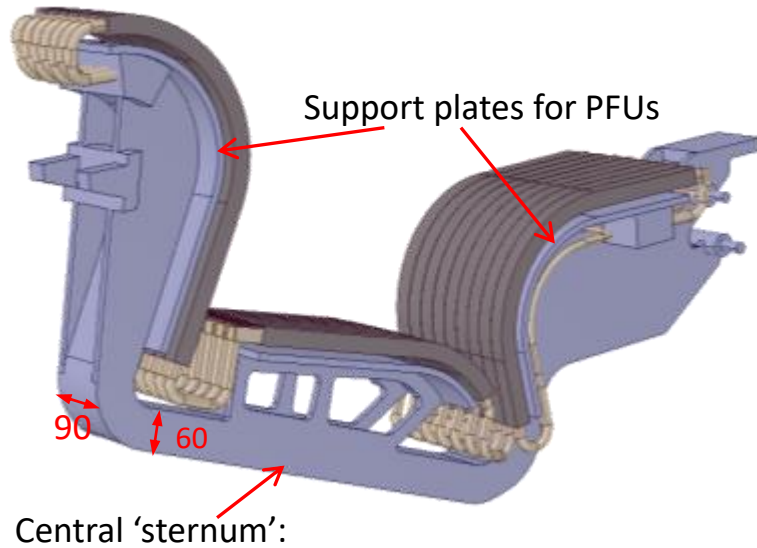
## The design

The **PFUs** connected in series externally to the **cassette** to reduce the **size of the cassette body**



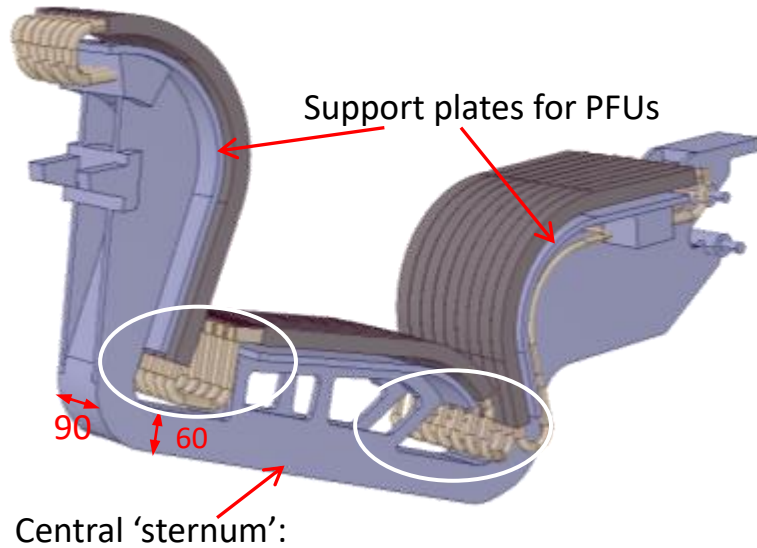
## The design

The **PFUs connected in series externally to the cassette** to reduce the **size of the cassette body**

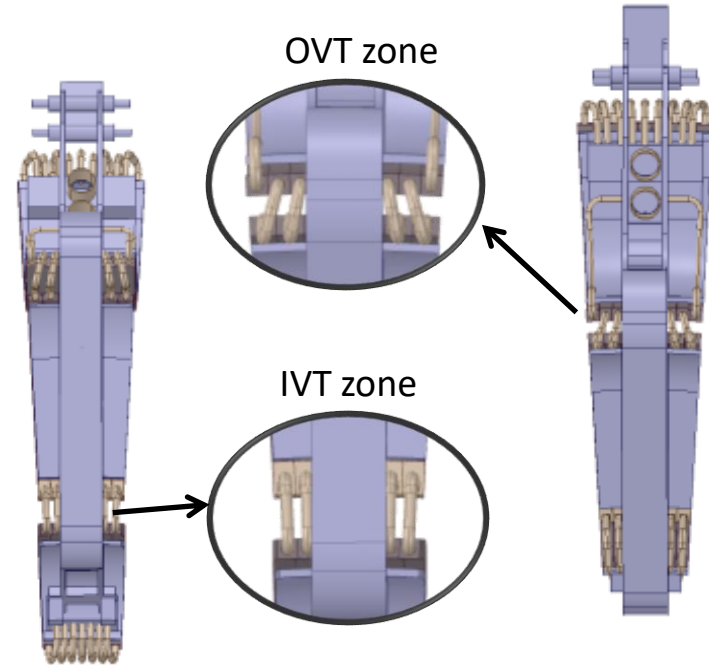


## The design

The **PFUs** connected in series externally to the **cassette** to reduce the **size of the cassette body**

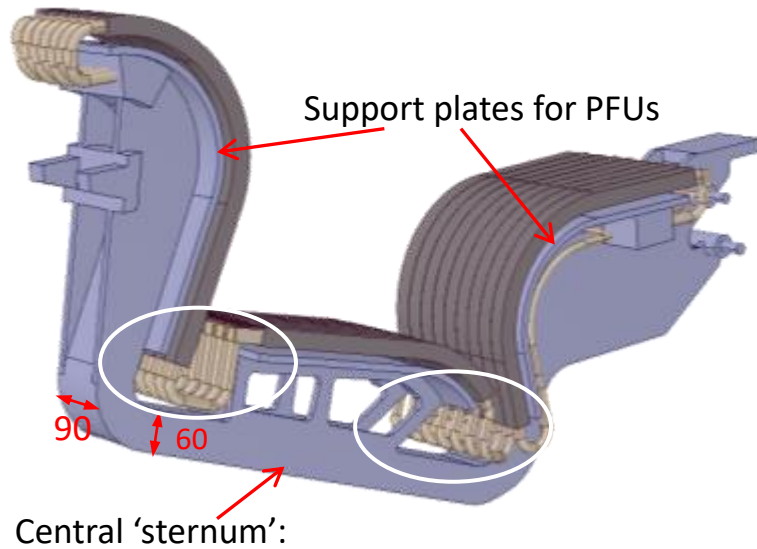


Slots for the pumping

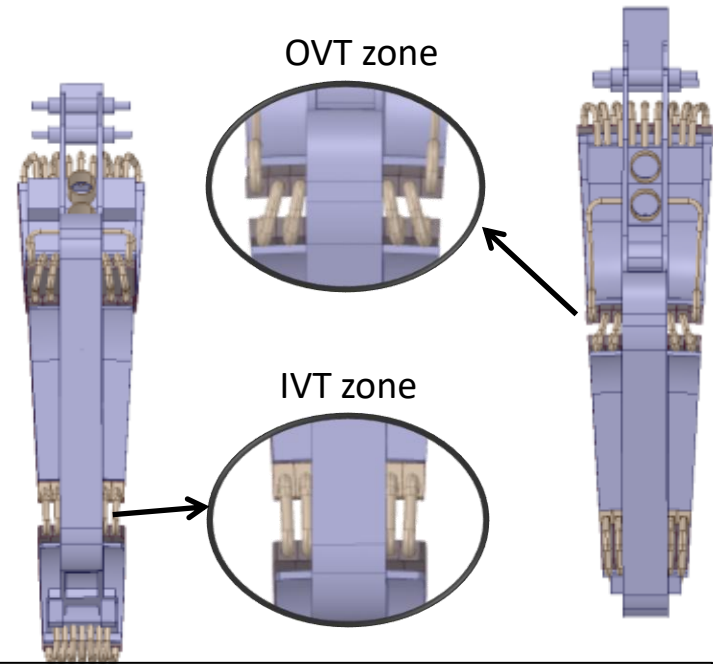


## The design

The **PFUs** connected in series externally to the **cassette** to reduce the **size of the cassette body**



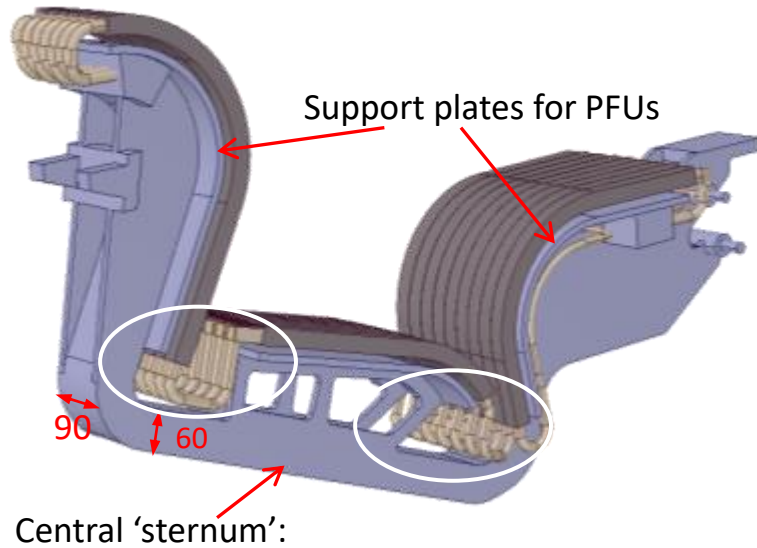
Slots for the pumping



3D simulations have demonstrated that the slots are adequate for SN configuration.

## The design

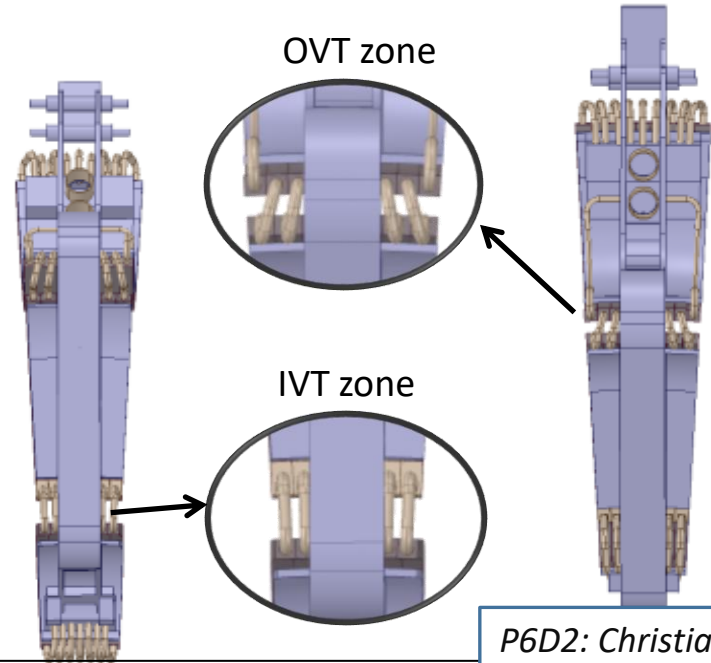
The **PFUs** connected in series externally to the **cassette** to reduce the **size of the cassette body**



Slots for the pumping

OVT zone

IVT zone



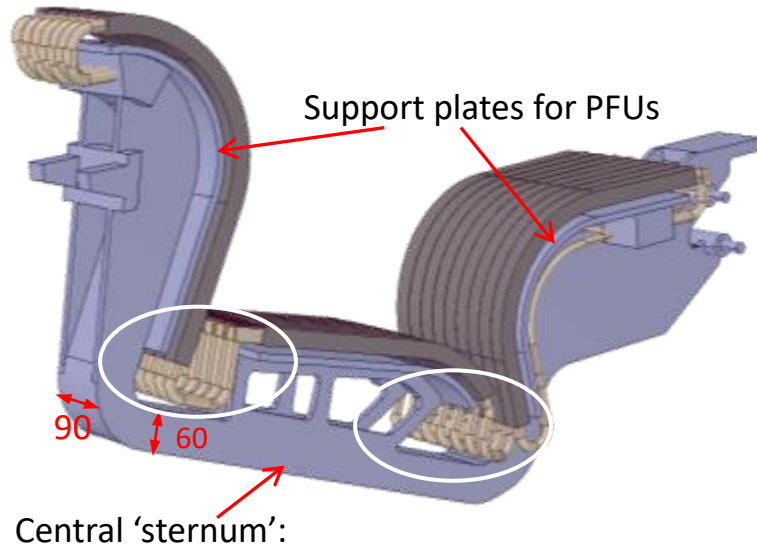
P6D2: Christian Day  
(on Thursday)

3D simulations have demonstrated that the slots are adequate for SN configuration.

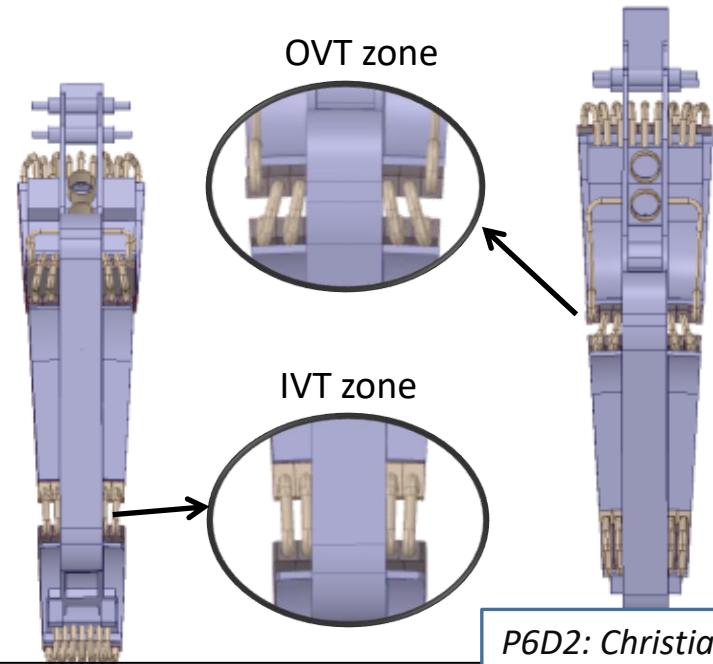


## The design

The **PFUs** connected in series externally to the **cassette** to reduce the **size of the cassette body**



Slots for the pumping



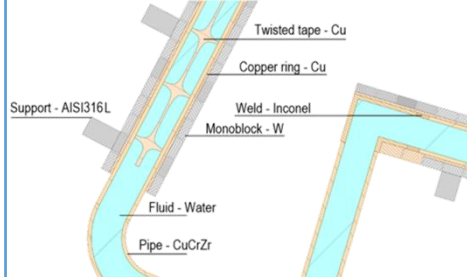
P6D2: Christian Day  
(on Thursday)

3D simulations have demonstrated that the slots are adequate for SN configuration. Design modifications are being studied to improve the pumping capacity and neutral compression for configurations with strike point on the Dome

## Thermo-fluid dynamics & mechanical analysis

### CFD Model:

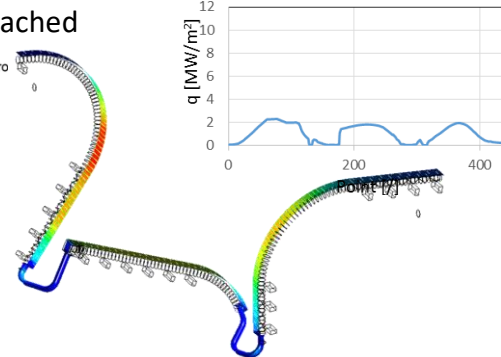
Single-phase fluid model  
Conjugate heat transfer  
k-w SST turbulence model  
Pipe roughness:  $5 \mu\text{m}$



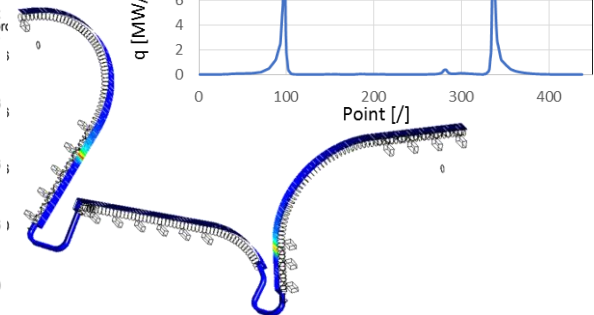
### Thermal load from edge simulations for various scenarios

#### SN Detached

Wall Heat Flux  
pipe\_heatflux\_pro  
 $2.285 \times 10^6$   
 $1.714 \times 10^6$   
 $1.143 \times 10^6$   
 $5.713 \times 10^5$   
 $0.000 \times 10^0$   
[W m<sup>-2</sup>]

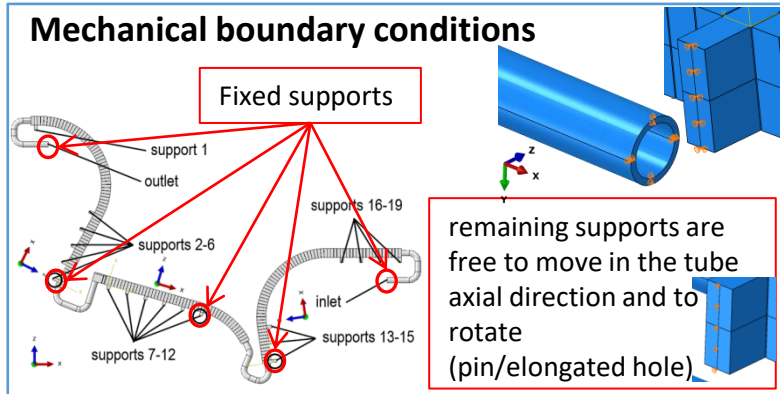


Wall Heat Flux  
pipe\_heatflux\_pr  
 $1.113 \times 10^7$   
 $8.347 \times 10^6$   
 $5.565 \times 10^6$   
 $2.782 \times 10^6$   
 $0.000 \times 10^0$   
[W m<sup>-2</sup>]



### Mechanical boundary conditions

Fixed supports

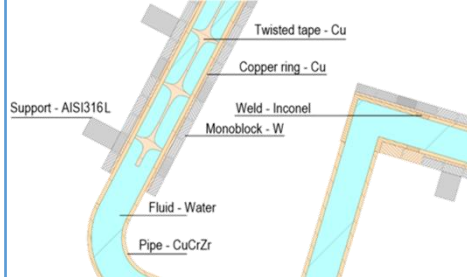


remaining supports are free to move in the tube axial direction and to rotate (pin/elongated hole)

## Thermo-fluid dynamics & mechanical analysis

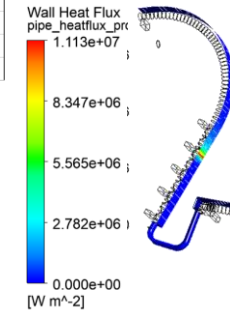
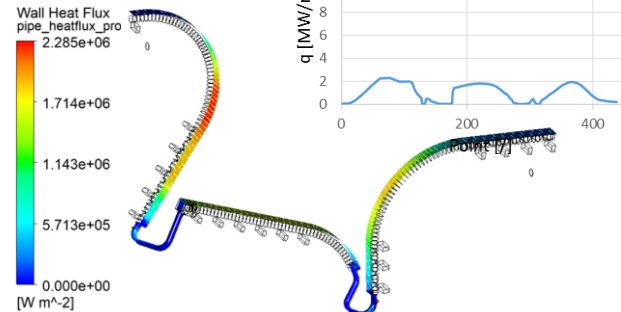
### CFD Model:

Single-phase fluid model  
 Conjugate heat transfer  
 k-w SST turbulence model  
 Pipe roughness: 5  $\mu\text{m}$

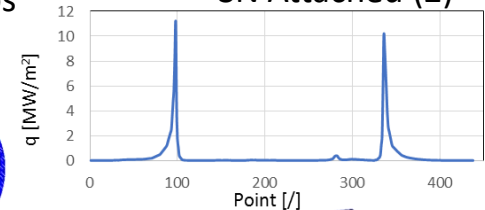


### Thermal load from edge simulations for various scenarios

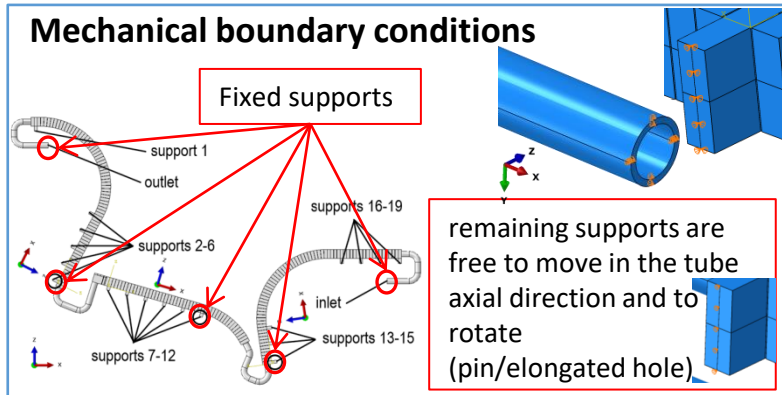
#### SN Detached



#### SN Attached (2)



### Mechanical boundary conditions



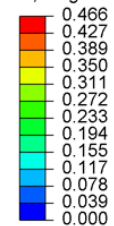
remaining supports are free to move in the tube axial direction and to rotate (pin/elongated hole)

### Results

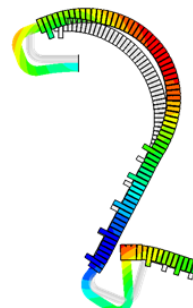
	PIPE (CuCrZr)		WELD (Inconel)		SUPPORTS (AISI 316)	
<b>SN Detached</b>						
MISES [MPa]	MAX	AVG	MAX	AVG	MAX	AVG
T [°C]	MAX	AVG	MAX	AVG	MAX	AVG
<b>SN Attached (2)</b>						
MISES [MPa]	MAX	AVG	MAX	AVG	MAX	AVG
T [°C]	MAX	AVG	MAX	AVG	MAX	AVG

CuCrZr @250°C  
 Sy=331Mpa (Treat. A)

U, Magnitude



Displacement  
 SN Detached

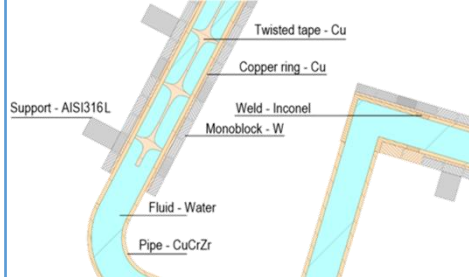


# Conceptual design: poloidal shaping

## Thermo-fluid dynamics & mechanical analysis

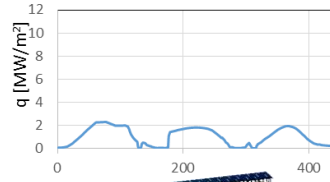
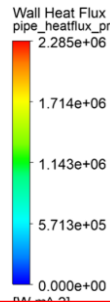
### CFD Model:

- Single-phase fluid model
- Conjugate heat transfer
- k-w SST turbulence model
- Pipe roughness: 5  $\mu\text{m}$

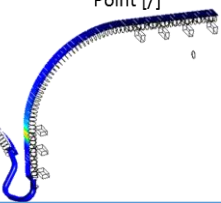
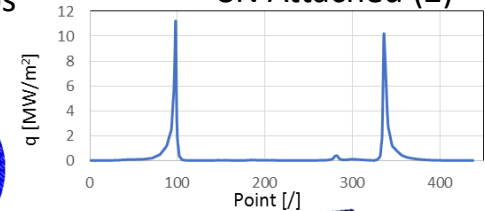


### Thermal load from edge simulations for various scenarios

#### SN Detached



#### SN Attached (2)

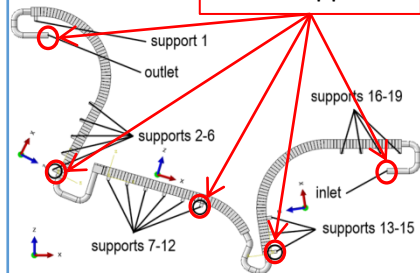


Verification under Electro-Magnetic loads is ongoing

CuCrZr @250°C  
 $S_y=331\text{Mpa}$  (Treat. A)

### Mechanical boundary conditions

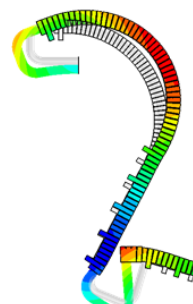
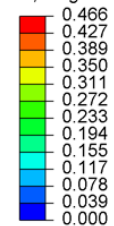
#### Fixed supports



remaining supports are free to move in the tube axial direction and to rotate (pin/elongated hole)

SN Detached		MAX	AVG	MAX	AVG	MAX	AVG
MISES [MPa]		171.32	67.45	130.20	67.31	175.75	23.36
T [°C]		126.29	74.79	131.08	90.73	84.00	70.13
SN Attached (2)		MAX	AVG	MAX	AVG	MAX	AVG
MISES [MPa]		215.77	50.71	71.84	41.85	149.45	20.26
T [°C]		222.54	63.53	63.08	62.17	89.39	63.68

#### U, Magnitude



Displacement  
 SN Detached